



Production of dissipative vortices by solid bodies
in incompressible fluid flows: comparison between
Prandtl, Navier-Stokes and Euler solutions

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IPAM, UCLA, October 1st 2014*

What is turbulence?

Turbulence is a state that **fluid flows** reach when they become **unstable** and highly **fluctuating**.

Hypotheses :

- The **fluid** is supposed to be a **continuous medium** when the observation scale is much larger than the mean free path of molecules,
- The **fluid flow** is supposed to be **incompressible**, *i.e.*, non-divergent.

Etymology of the word 'turbulence' :

turba-ae, crowd, mob

turbo-inis, vortex

A mob of vortices interacting together
on a wide range of temporal and spatial scales.

Fluid flows reach the **fully-developed turbulent regime** when they become **highly mixing**.

1.

**History of
d'Alembert's paradox**

Jean Le Rond d'Alembert
(1717-1783)



Leonhard Euler
(1707-1783)



Mathematical Prize 1750

On 16th May 1748 the Prussian Academy of Sciences, presided by Euler, offered a prize to the mathematician who could propose a:

'Theoria resistentiae quam patitur corpus in fluido motum, ex principiis omnino novis et simplissimis deducta, habita ratione tum velocitatis, figurae, et massae corporis moti, tum densitatis & compressionis partium fluidi'.

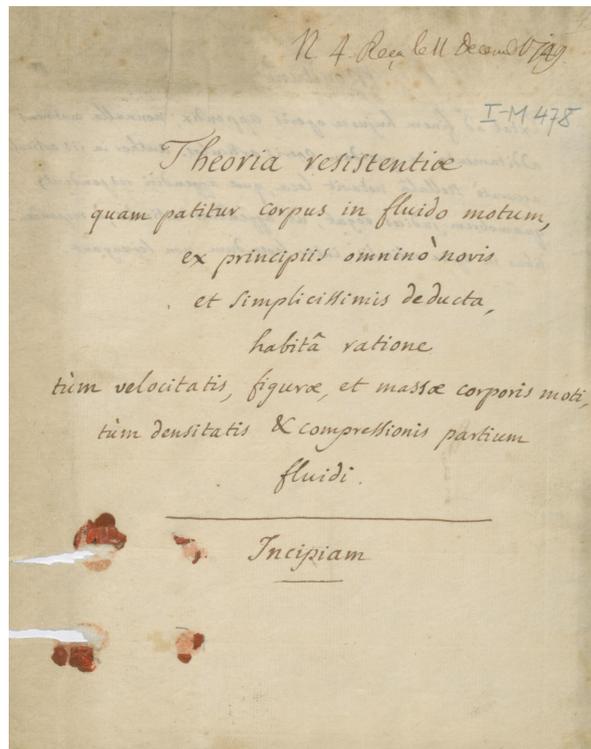
On 25th November 1749 d'Alembert sent a 137 pages manuscript, but Euler decided to postpone the prize to 1752.

Grimberg, D'Alembert et les équations aux dérivées partielles en hydrodynamique, Thèse de Doctorat, Université de Paris VII, 1998

D'Alembert's theory of fluid resistance

D'Alembert was upset and took back his manuscript of 1749. He translated it into French and published it in 1752 under the title '*Essai d'une nouvelle théorie de la résistance des fluides*'.

1749



1752

ESSAI
D'UNE
NOUVELLE THEORIE
DE LA
RÉSISTANCE DES FLUIDES.
Par M. D'ALEMBERT, de l'Académie Royale des Sciences
de Paris, de celle de Prusse, & de la Société Royale de Londres.



A PARIS,
Chez DAVID l'aîné, Libraire, rue S. Jacques, à la Plume d'or.
M D C C L I I.
AVEC APPROBATION ET PRIVILEGE DU ROI.

The prize was finally given in 1752 to Jacob Adami, a friend of Euler, and published by the Prussian Academy.

D'Alembert's paradox

Euler had already noticed the fact that potential flow exerts no drag on moving bodies in a work he published in 1745 on 'New principles of gunnery'.

While working on the Berlin Academy Prize, d'Alembert was also conscious of that problem and wrote:

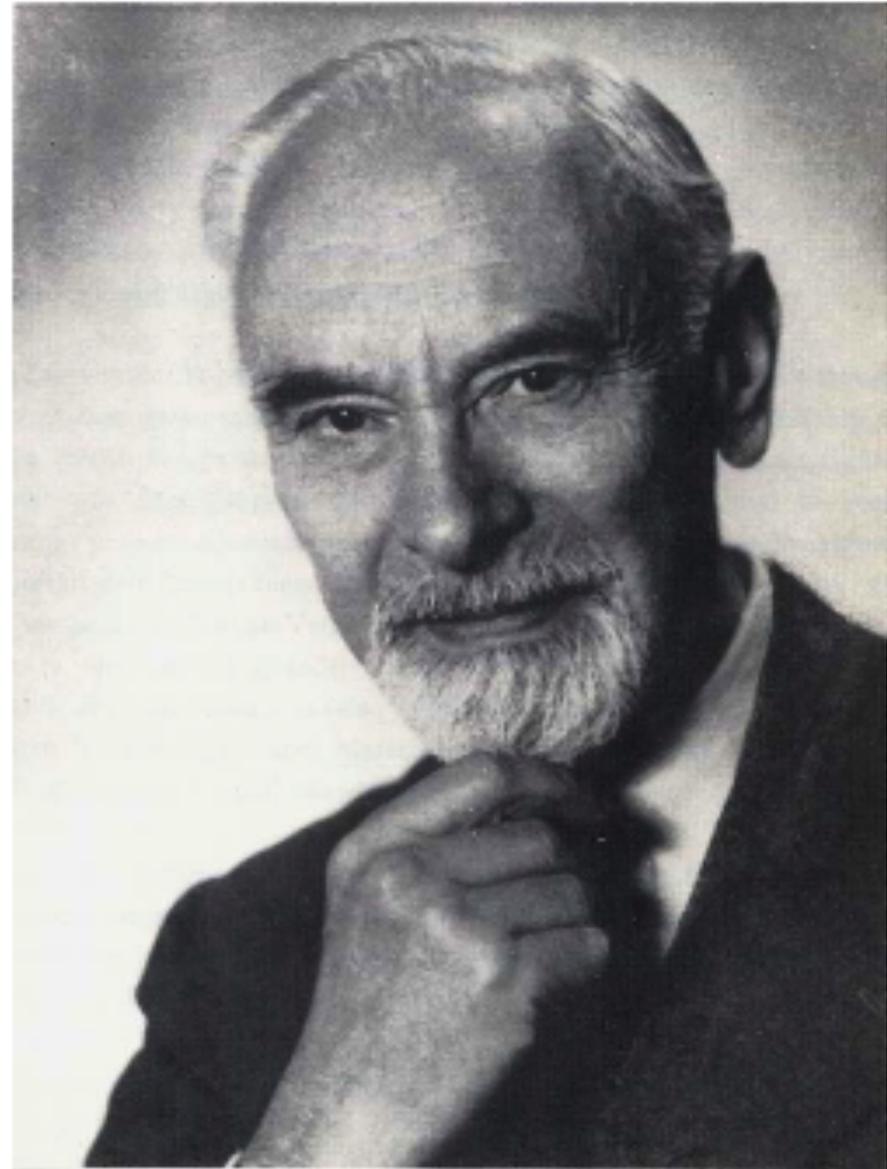
*'It seems to me that the theory, developed in all possible rigor, gives, at least in several cases, a **strictly vanishing resistance**, a singular paradox which I leave to future geometers to elucidate.'*

Darrigol, World of flows: a history of hydrodynamics from Bernoulli to Prandtl, Oxford university Press, 2005

*Adhémar Jean-Claude
Barré de Saint-Venant
(1797-1886)*



*Ludwig Prandtl
(1875-1953)*



Resolution proposed by Saint-Venant

In 1846 he wrote a note to the 'Société Philomatique', published later by the 'Académie des Sciences', stating that:

'But one finds another result if, instead of an inviscid fluid, object of the calculation of the geometers of the last century, one uses a real fluid, composed of a finite number of molecules and exerting in its state of motion unequal pressure forces having components tangential to the surface elements through which they act; components to which we refer as the friction of the fluid, a name which has been given to them since Descartes and Newton until Venturi.'

Saint-Venant, *Résistance des fluides: considérations historiques, physique et pratiques relatives au problème de l'action dynamique mutuelle d'un fluide à un solide, dans l'état de permanence supposé acquis par leurs mouvements*, Mémoires de l'Académie des sciences, 44, 1-280, 1888

Resolution proposed by Prandtl

At the 3rd ICM conference held in 1904 in Heidelberg, Prandtl proposed a theory based on the hypothesis:

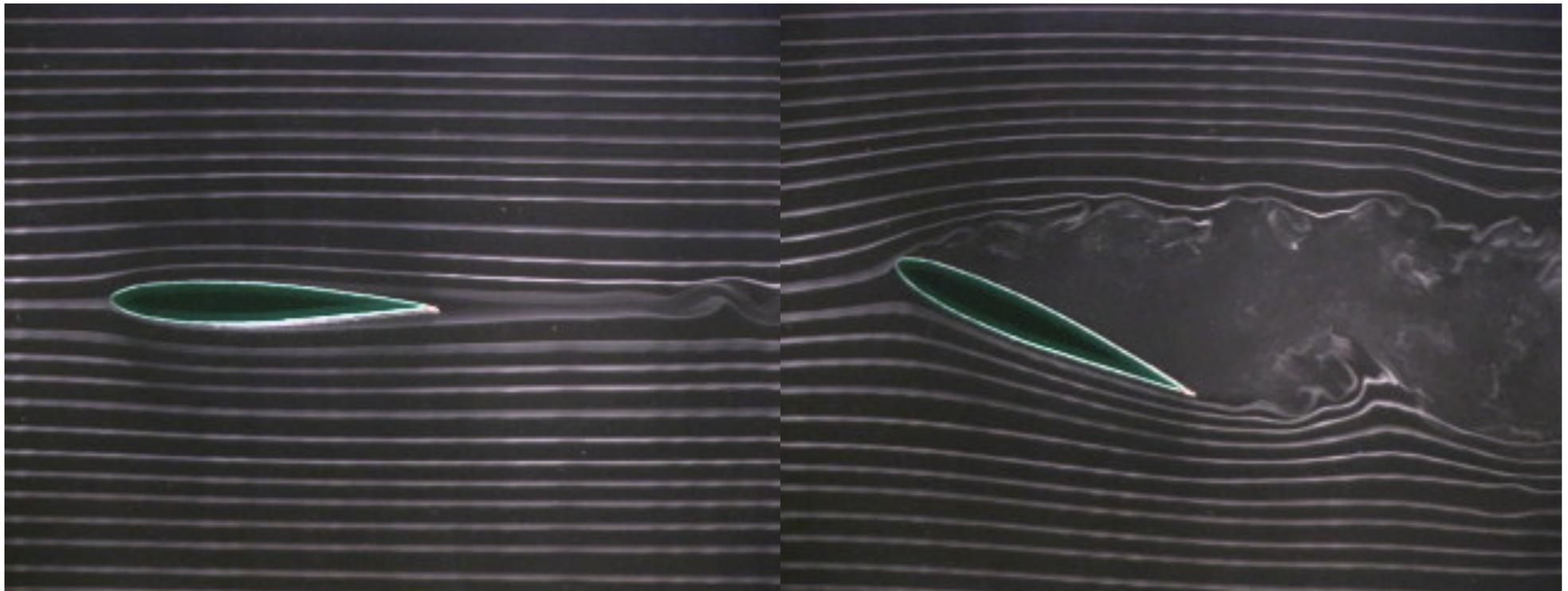
‘The viscosity is supposed to be so small that it can be disregarded wherever there are no great velocity differences. [...] The most important aspect of the problem is the behavior of the fluid on the surface of the solid body. [...] In the thin transition layer, the great velocity differences will [...] produce noticeable effects in spite of the small viscosity constants. [...] It is therefore possible to pass to the limit $\nu = 0$ and still retain the same flow figure.’

Inviscid limit = Euler equation + Prandtl viscous equation

*Prandtl, NACA YM-342,
English translation, 1927*

Boundary layer theory

- Prandtl was aware that his approach is only valid if the boundary layer remains attached to the wall (*left*), *i.e.*, away from separation points.
- Separated flow regions, *i.e.*, where the boundary layer detaches (*right*), have to be included « by hand » since Prandtl's theory doesn't predict their behavior.



2.

**Inviscid limit of the
Navier-Stokes equations**

What is the inviscid limit of Navier-Stokes?

Navier-Stokes equations with no-slip boundary conditions:

$$(NS) \begin{cases} \partial_t \mathbf{u} + (\mathbf{u} \cdot \nabla) \mathbf{u} = -\nabla p + \frac{1}{Re} \nabla^2 \mathbf{u} \\ \nabla \cdot \mathbf{u} = 0 \\ \mathbf{u}|_{\partial\Omega} = \mathbf{0}, \quad \mathbf{u}(0, \cdot) = \mathbf{v} \end{cases}$$

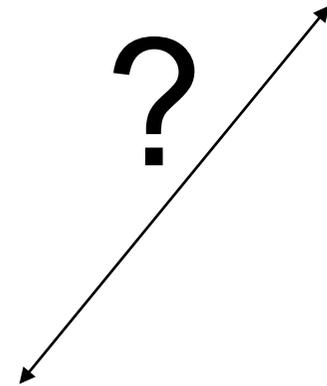

 Navier-Stokes solutions $\mathbf{u}_{Re}(t, \mathbf{X})$ for $\nu \rightarrow 0$ and $Re \rightarrow +\infty$

The Reynolds number $Re = U / \nu$ appears when non dimensional quantities are introduced.

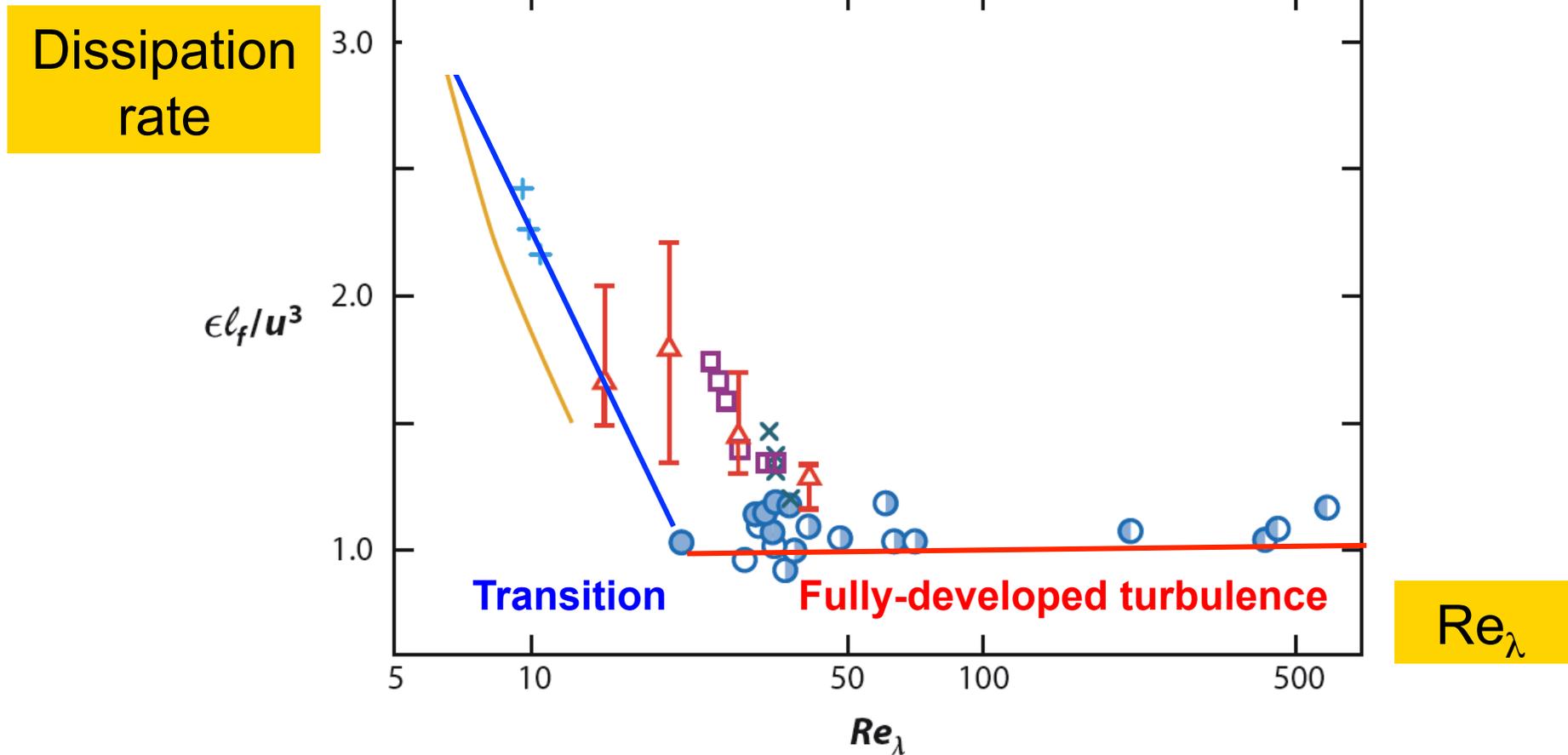
Euler equations with slip b.c.:

$$(E) \begin{cases} \partial_t \mathbf{u} + (\mathbf{u} \cdot \nabla) \mathbf{u} = -\nabla p \\ \nabla \cdot \mathbf{u} = 0 \\ \mathbf{u}|_{\partial\Omega} \cdot \mathbf{n} = 0, \quad \mathbf{u}(0, \cdot) = \mathbf{v} \end{cases}$$


 Euler's solutions $\mathbf{u}(t, \mathbf{X})$ for $\nu = 0$ and $Re = +\infty$



Dissipation rate: laboratory experiments



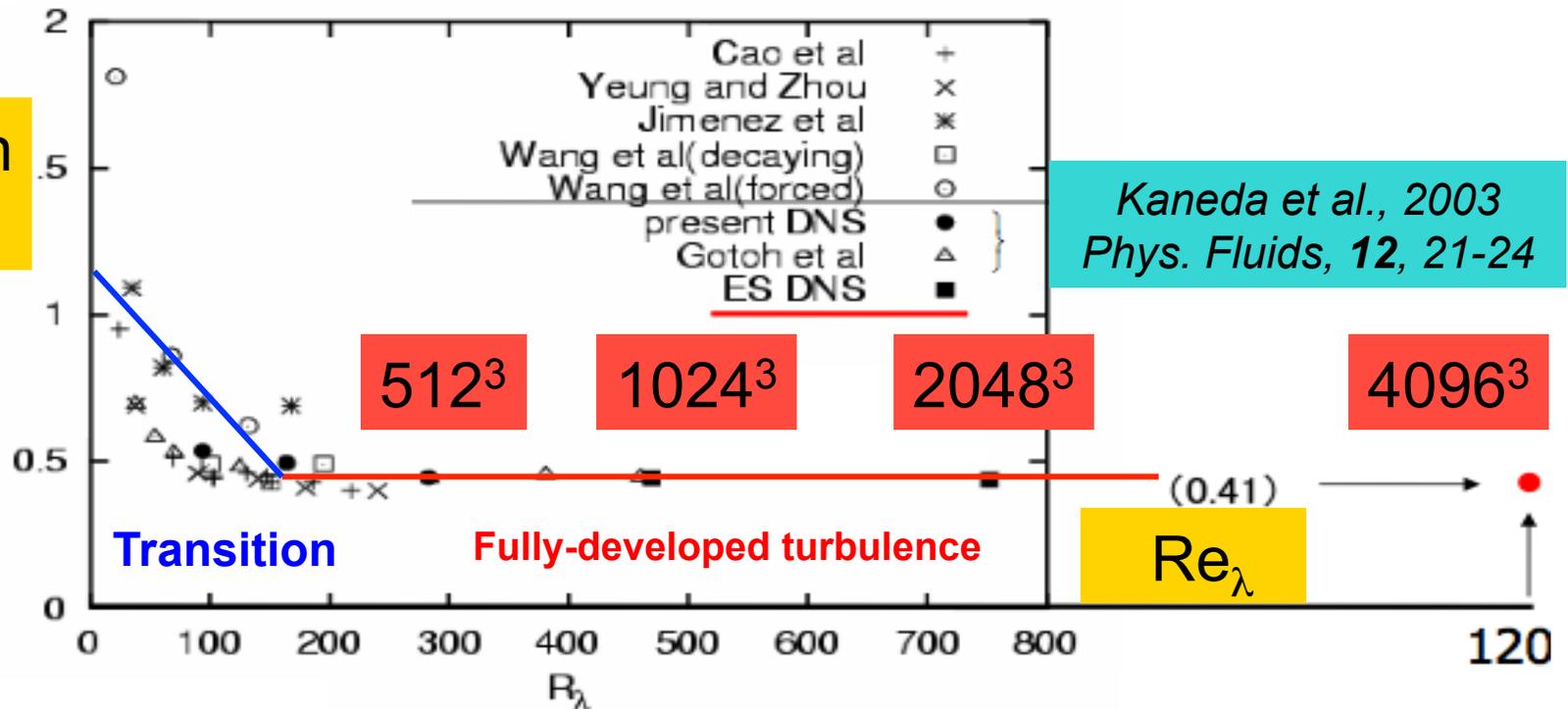
Vassilicos, *Ann. Rev. Fluid Mech.*, **47**, 2015

Dissipation rate : numerical experiments

Normalized energy dissipation $\rightarrow ?$
 as $\nu \rightarrow 0$, or $Re \rightarrow \infty$

$$\epsilon L / u'^3$$

Dissipation rate



Both laboratory experiments and numerical experiments of turbulent flows show that the dissipation rate becomes independent of the fluid viscosity

Dissipation of energy in the inviscid limit

- In an incompressible flow ($\rho = 1$)

$$\frac{dE}{dt} = \frac{d}{dt} \int \frac{\mathbf{u}^2}{2} = -\nu \int \omega^2 = -2\nu Z$$

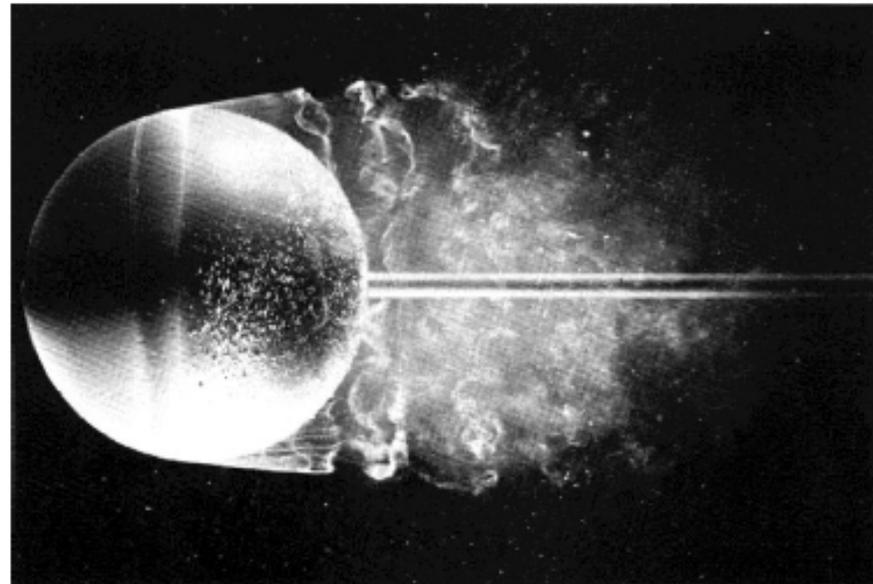
- To dissipate energy, vorticity needs to be **created** and/or **amplified**, in such a way that $Z \sim \nu^{-1}$.

Possible vorticity distributions:

$\omega \sim \nu^{-1/2}$ over $O(1)$ area,

$\omega \sim \nu^{-1}$ over $O(\nu)$ area.

E energy, Z enstrophy,
 ν fluid kinematic viscosity
 ω flow vorticity.



Why is dissipation of energy so essential ?

- Kato (1984) proved (roughly stated):

The NS solution converges towards the Euler solution in L^2 :

$$\forall t \in [0, T], \|u_{\text{Re}}(t) - u(t)\|_{L^2(\Omega)} \xrightarrow{\text{Re} \rightarrow \infty} 0,$$

if and only if

the energy dissipation during this interval vanishes:

$$\Delta E_{\text{Re}}(0, T) = \text{Re}^{-1} \int_0^T dt \int_{\Omega} d\mathbf{x} |\nabla \mathbf{u}(t, \mathbf{x})|^2 \xrightarrow{\text{Re} \rightarrow \infty} 0,$$

and even if and only if

it vanishes in a strip of width prop to Re^{-1} around the solid:

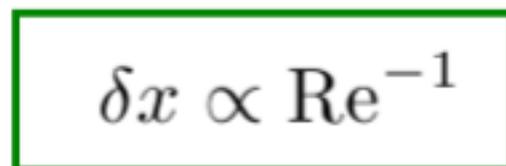
$$\text{Re}^{-1} \int_0^T dt \int_{\Gamma_{c\text{Re}^{-1}}} d\mathbf{x} |\nabla \mathbf{u}(t, \mathbf{x})|^2 \xrightarrow{\text{Re} \rightarrow \infty} 0, \quad \Gamma_{c\text{Re}^{-1}} = \left\{ \mathbf{x} \mid d(\mathbf{x}, \partial\Omega) < c\text{Re}^{-1} \right\}.$$

An important practical consequence

- To have any chance of observing energy dissipation (i.e. default of convergence towards the Euler solution), we need a smaller grid than Prandtl's (1904) prediction for attached boundary layers:

$$\delta x \propto \text{Re}^{-\frac{1}{2}}$$



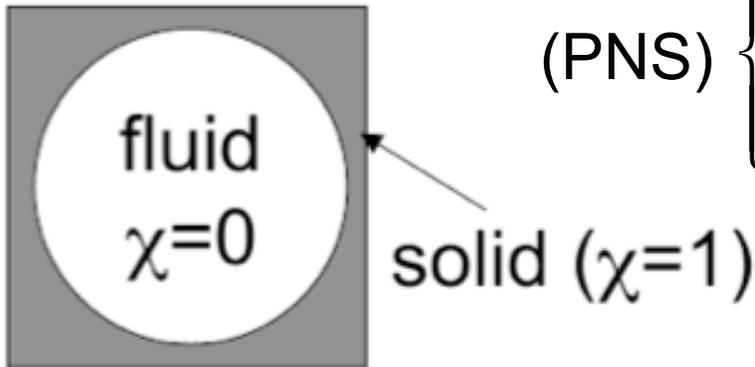

$$\delta x \propto \text{Re}^{-1}$$


3.

**Numerical
experiments**

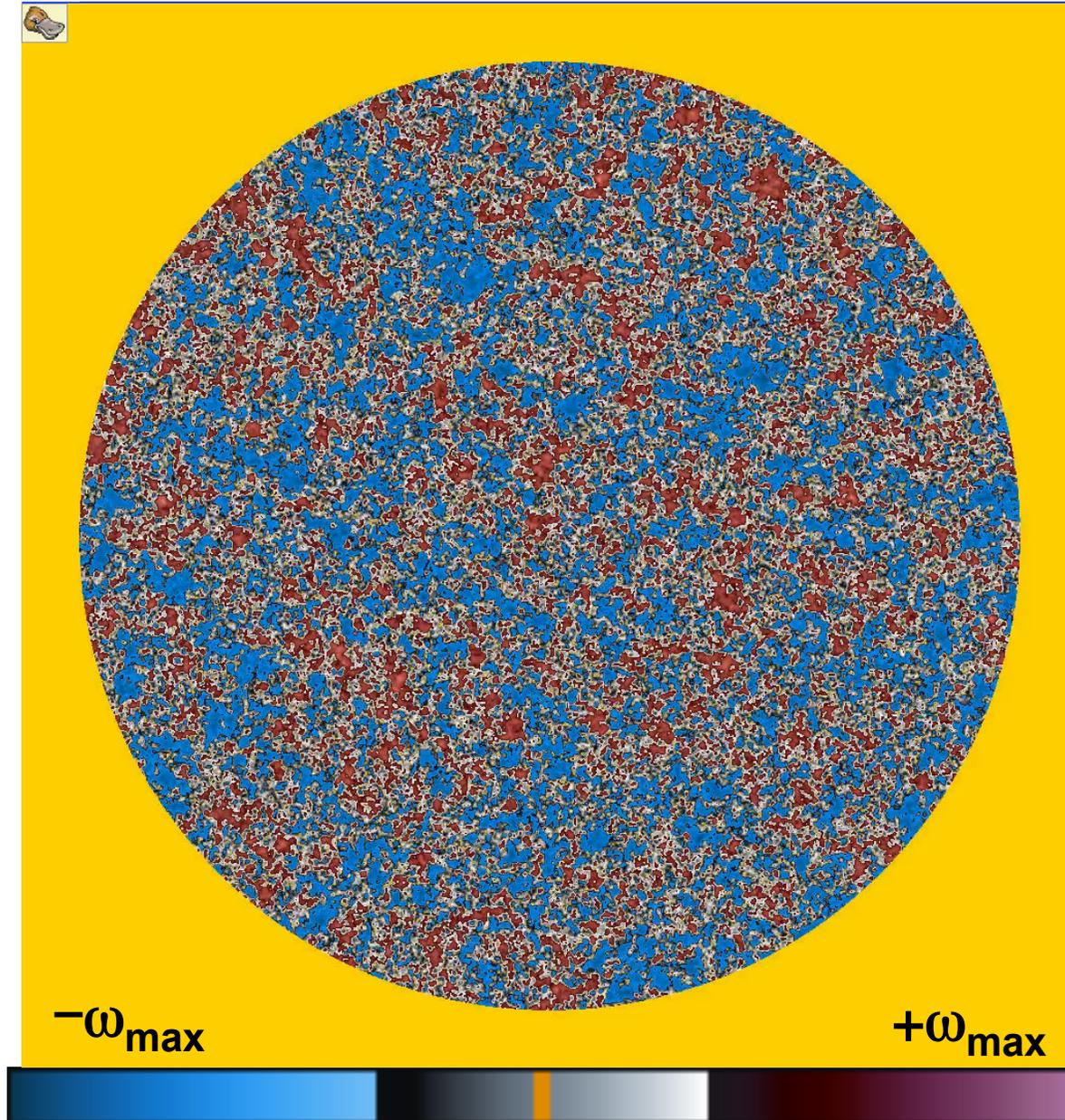
Volume penalization method to compute NS

- For efficiency and simplicity, we would like to stick to a **spectral solver in periodic, cartesian coordinates**.
- as a counterpart, we need to **add an additional term in the equations to approximate the effect of the boundaries**,
- the **geometry is encoded in a mask function χ** ,


$$(PNS) \begin{cases} \partial_t \mathbf{u} + (\mathbf{u} \cdot \nabla) \mathbf{u} = -\nabla p + \frac{1}{\text{Re}} \Delta \mathbf{u} - \frac{1}{\eta} \chi_0 \mathbf{u} \\ \nabla \cdot \mathbf{u} = 0, \quad \mathbf{u}(0, \mathbf{x}) = \mathbf{v} \end{cases}$$

solution $\rightarrow \mathbf{u}_{\text{Re}, \eta}$

31. Wall-bounded 2D turbulent flow

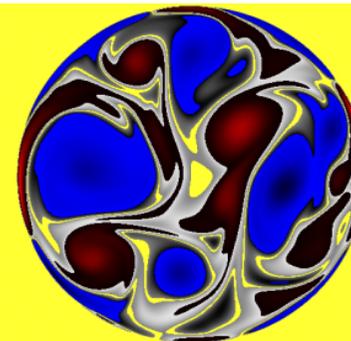
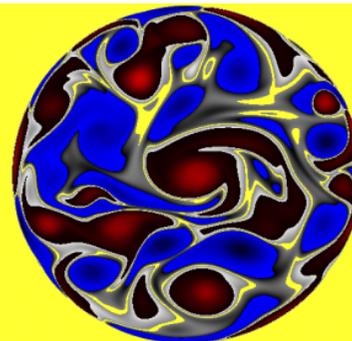
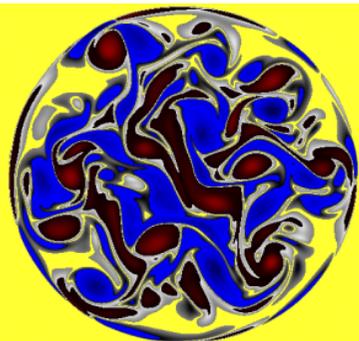
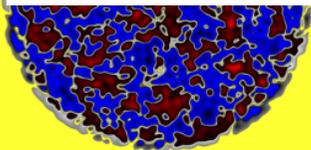


DNS
Resolution
 $N=1024^2$

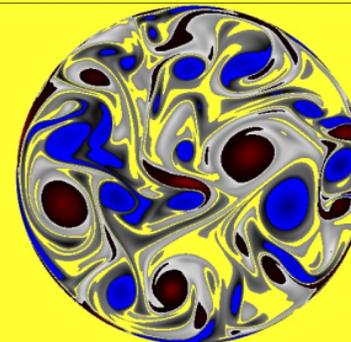
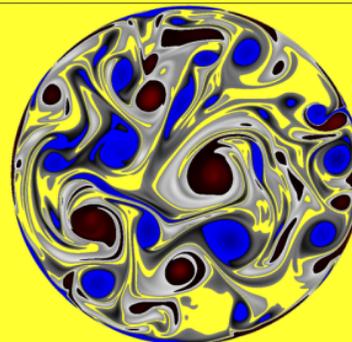
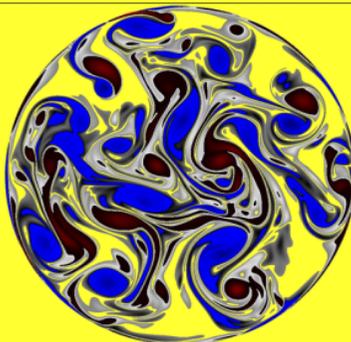
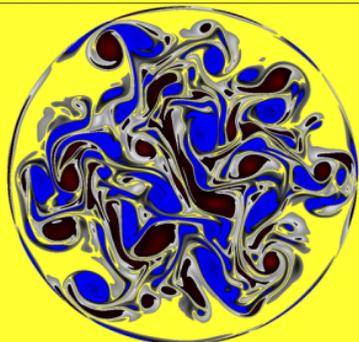
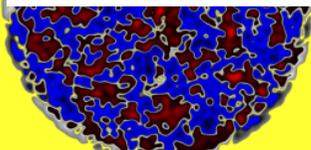
*Dealiased
pseudo-spectral
In space
and 3rd order
Runge-Kutta
In time*

K. Schneider and M. F.,
Phys. Rev. Lett., **95**,
244502 (2005)

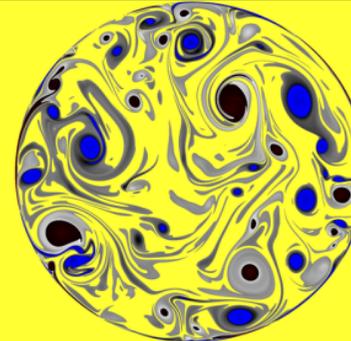
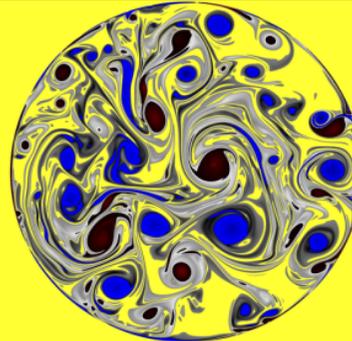
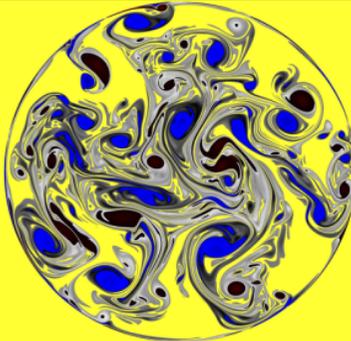
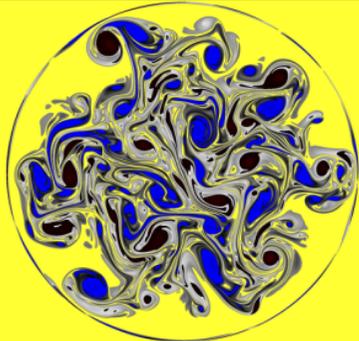
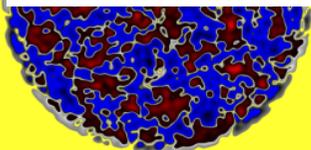
$N=1024^2$
 $Re=2 \cdot 10^3$



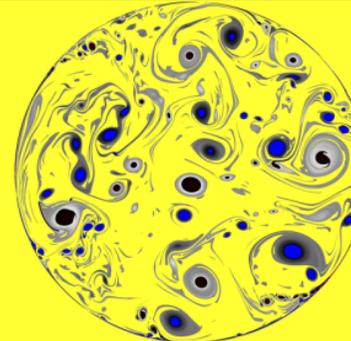
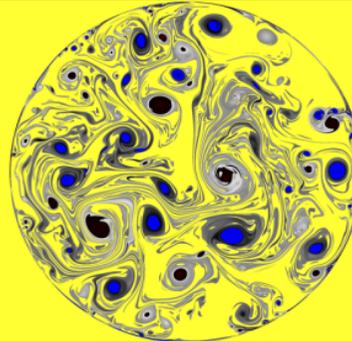
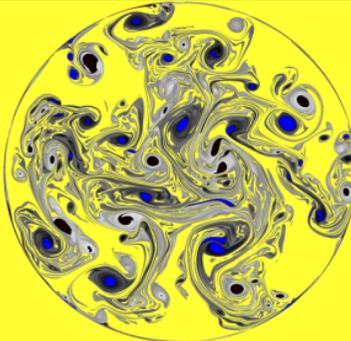
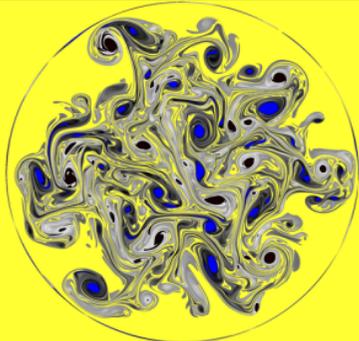
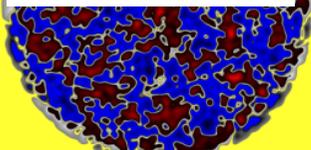
$N=2048^2$
 $Re=7 \cdot 10^3$



$N=4096^2$
 $Re=2 \cdot 10^4$

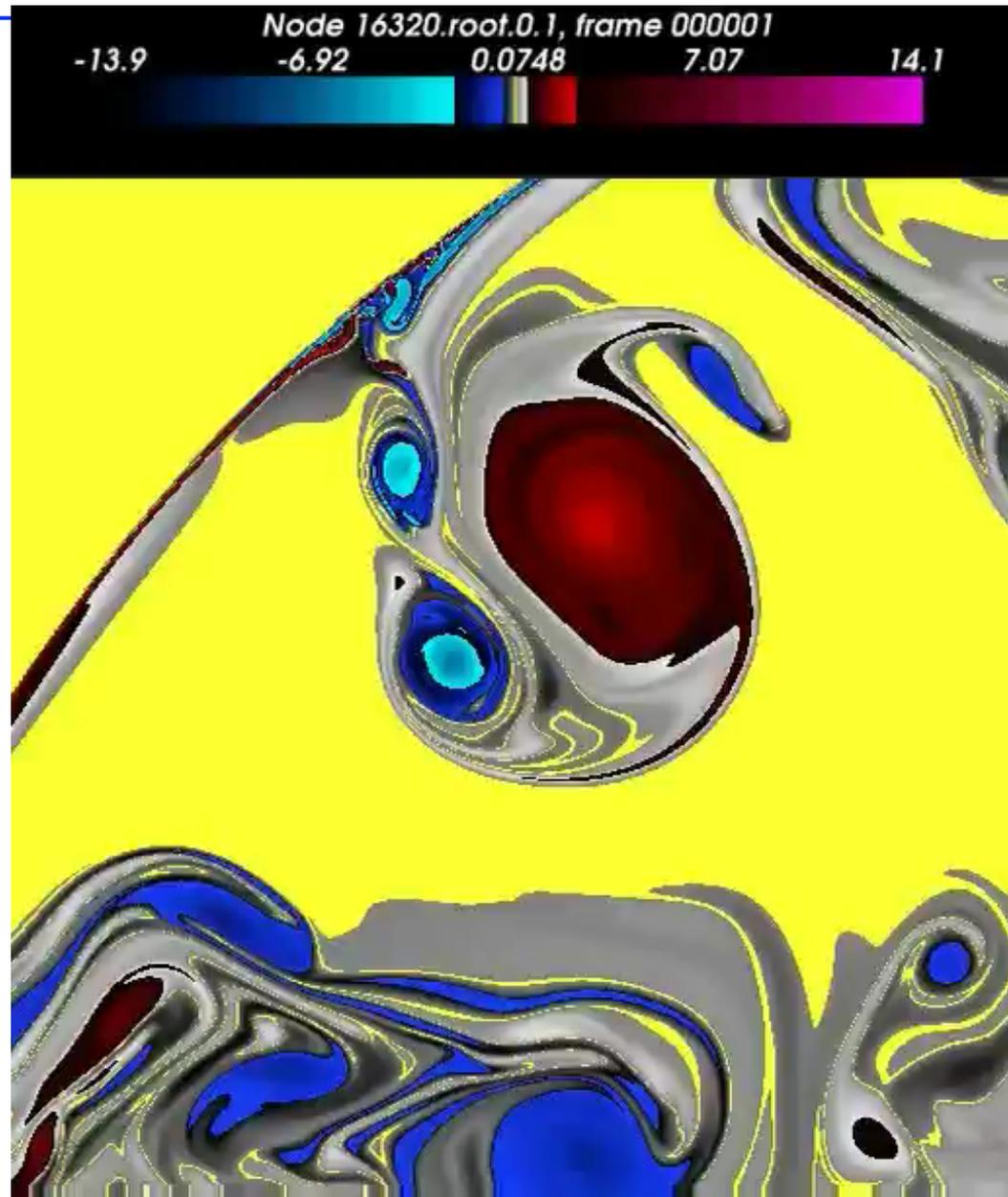


$N=8192^2$
 $Re=10^5$



Flow interaction with the wall

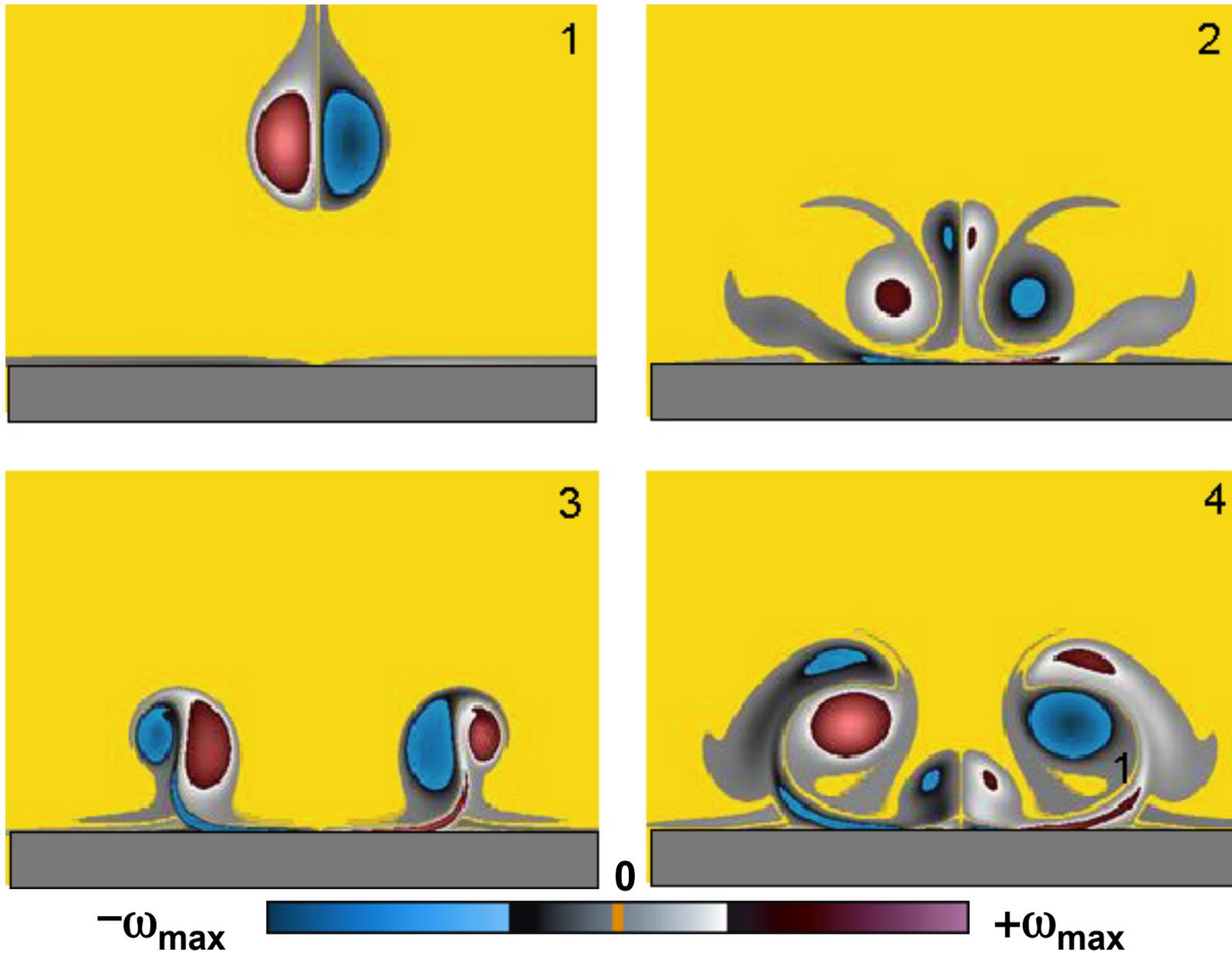
Resolution
 $N=8192^2$



*Time evolution
of vorticity
at the wall
computed on
IBM Blue-Gene,
IDRIS, 2010
(100 Tflops)*

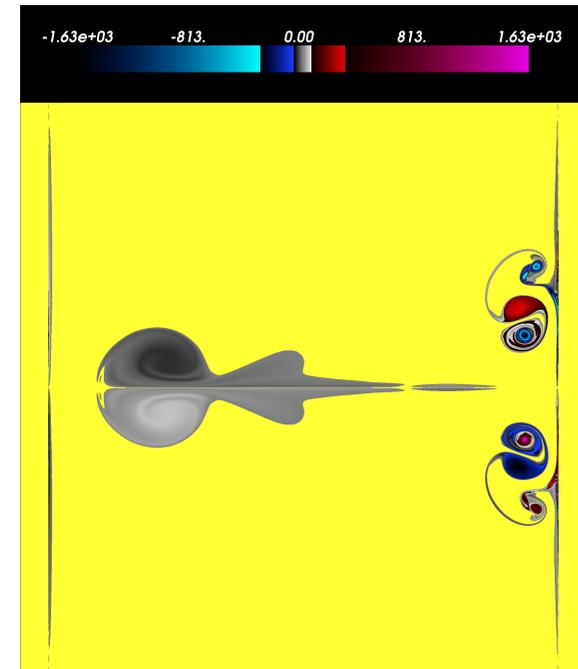
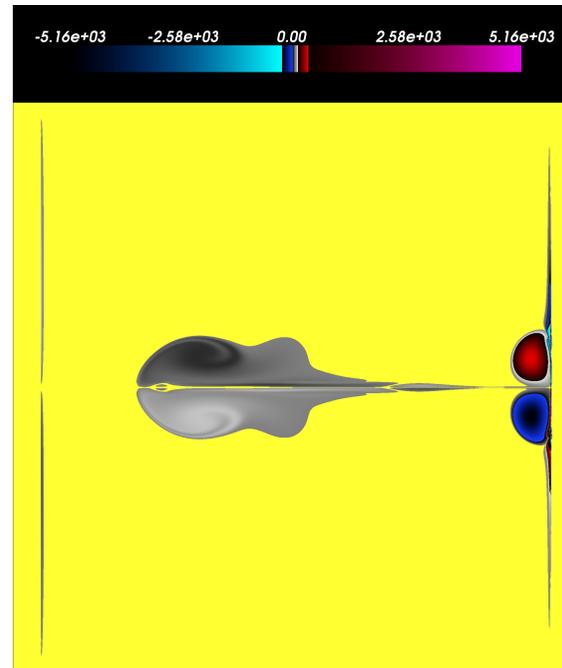
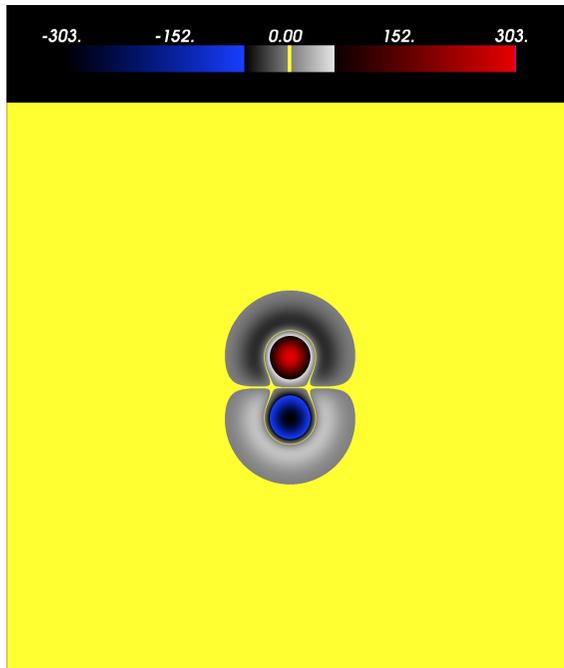
*Nguyen van yen,
M. F. and
Schneider,
2010*

32. Dipole crashing onto a wall



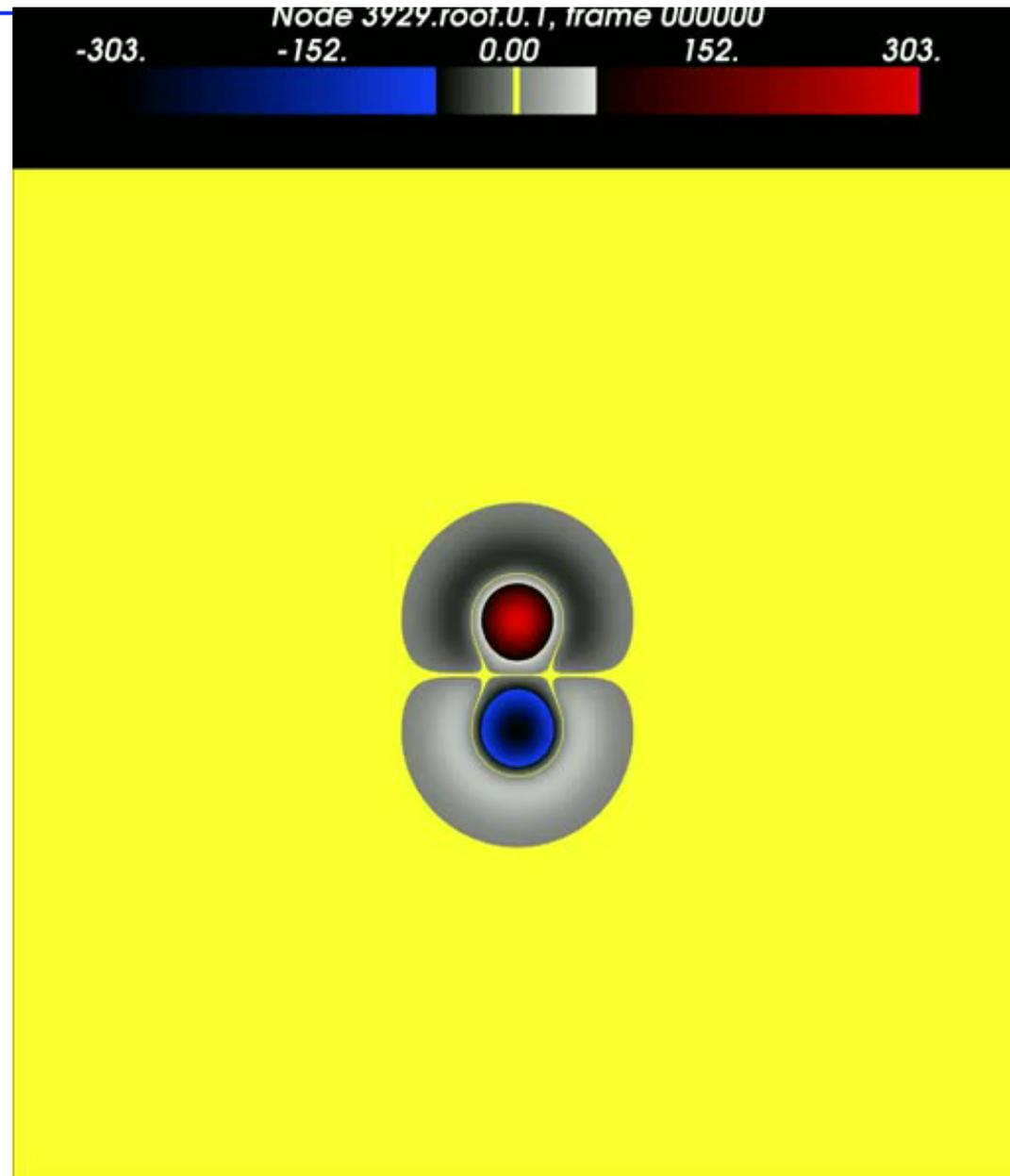
Dipole crashing onto a wall at $Re=8000$

DNS
Resolution
 $N=8192^2$



Dipole crashing onto a wall at $Re=8000$

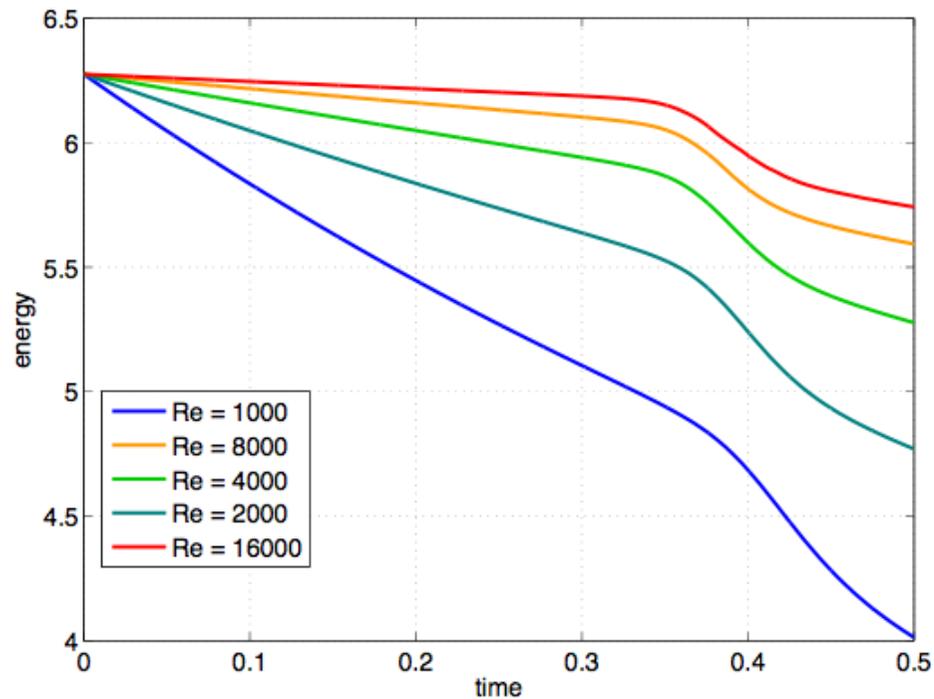
DNS
Resolution
 $N=8192^2$



Energy evolution

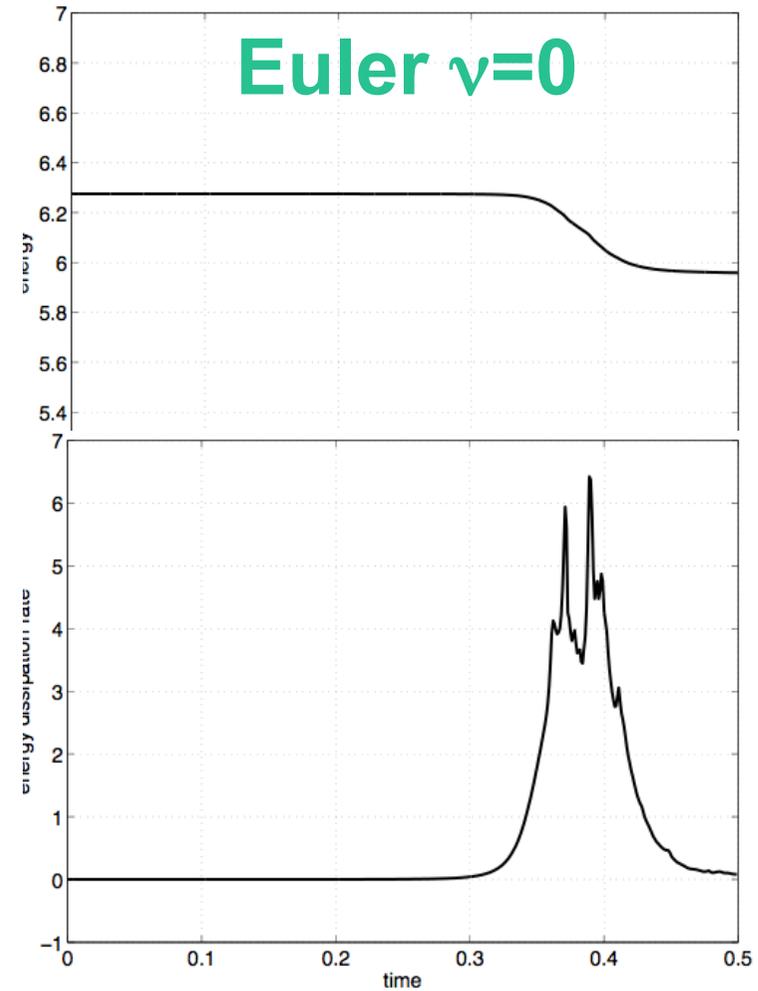
DNS
Resolution
 $N=8192^2$

Navier-Stokes
 $\nu > 0$



Time evolution of energy

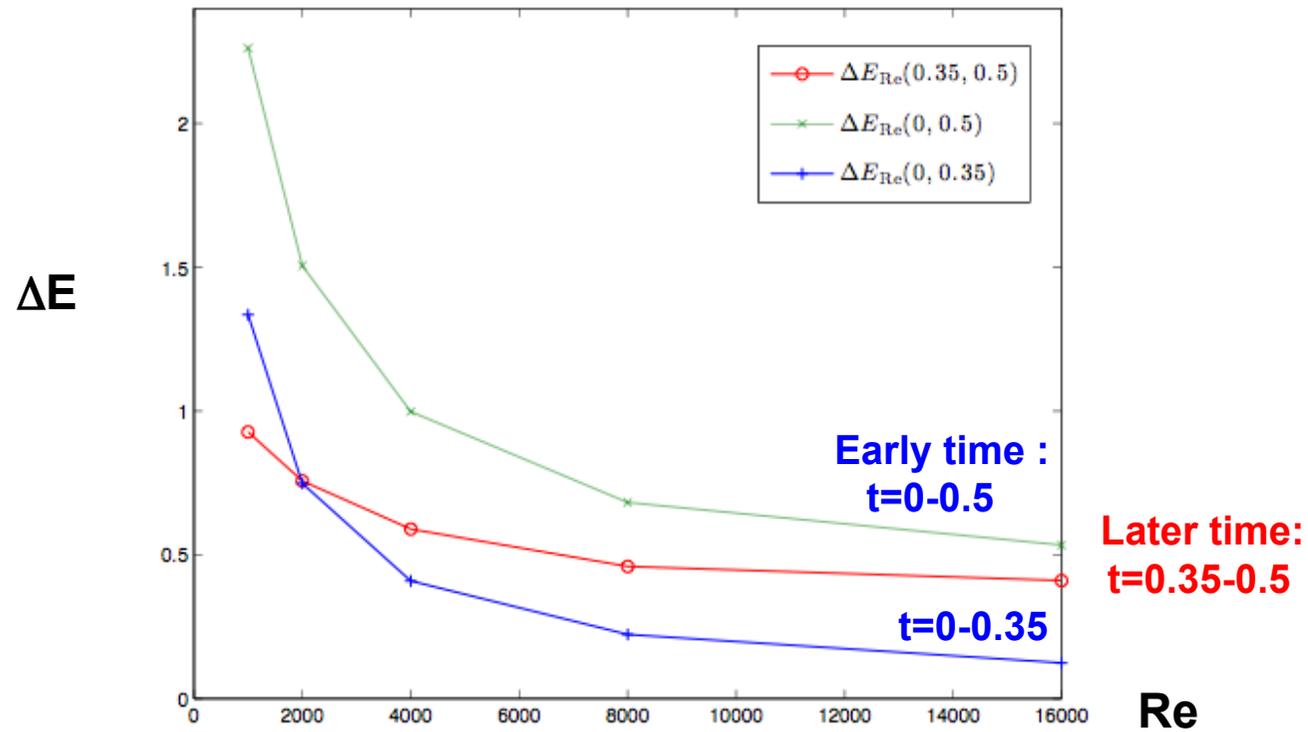
Time evolution of energy



Evolution of energy dissipation rate

Energy dissipation

Energy dissipated
when the dipole crashes onto the wall
at increasing Reynolds numbers



Dissipative structures

- Our experiments with the dipole crashing onto a wall suggest that the flow remains dissipative in the inviscid limit,
- it is tempting to relate these structures to energy dissipation,
- the kinetic energy density $e = \frac{|\mathbf{u}|^2}{2}$ obeys:

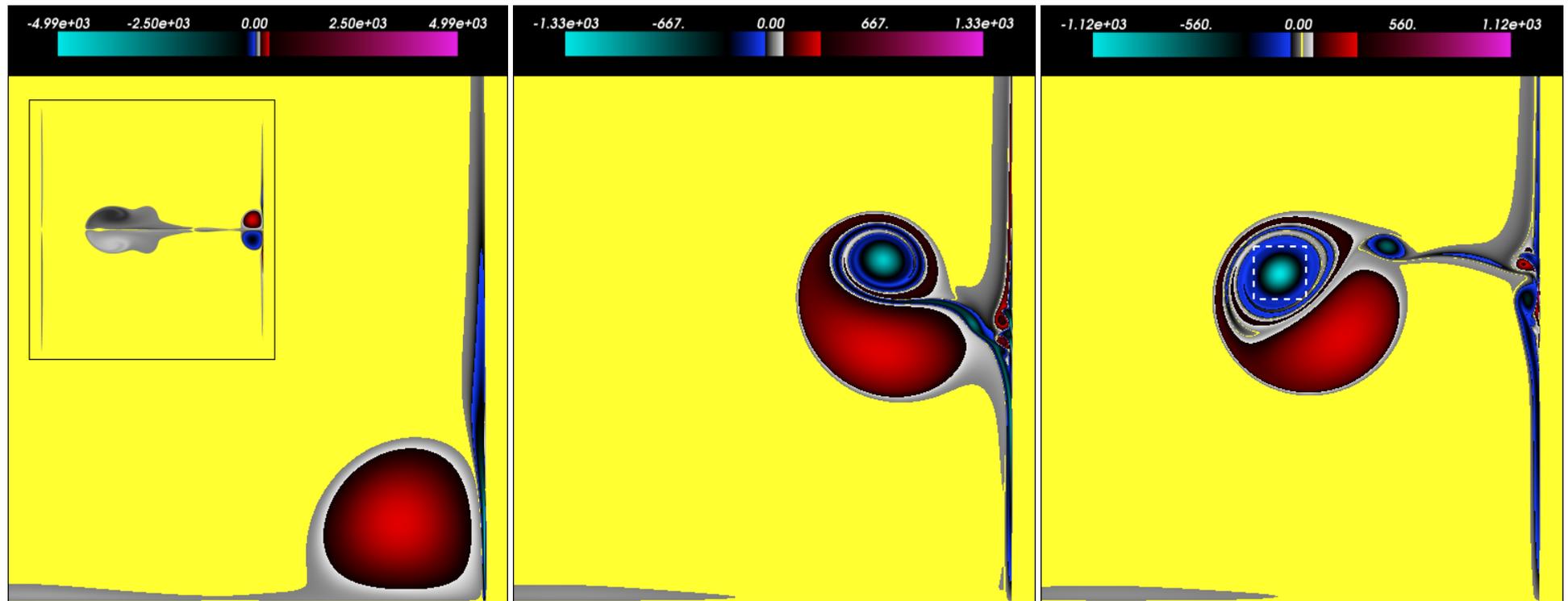
$$\partial_t e + \mathbf{u} \cdot \nabla(e + p) = \nu \Delta e - \nu |\nabla \mathbf{u}|^2$$

Local dissipation rate

DNS of dipole crashing onto a wall

Resolution
 $N=16384^2$

Nguyen van yen, M. F.
and Schneider,
PRL, **106**(18)



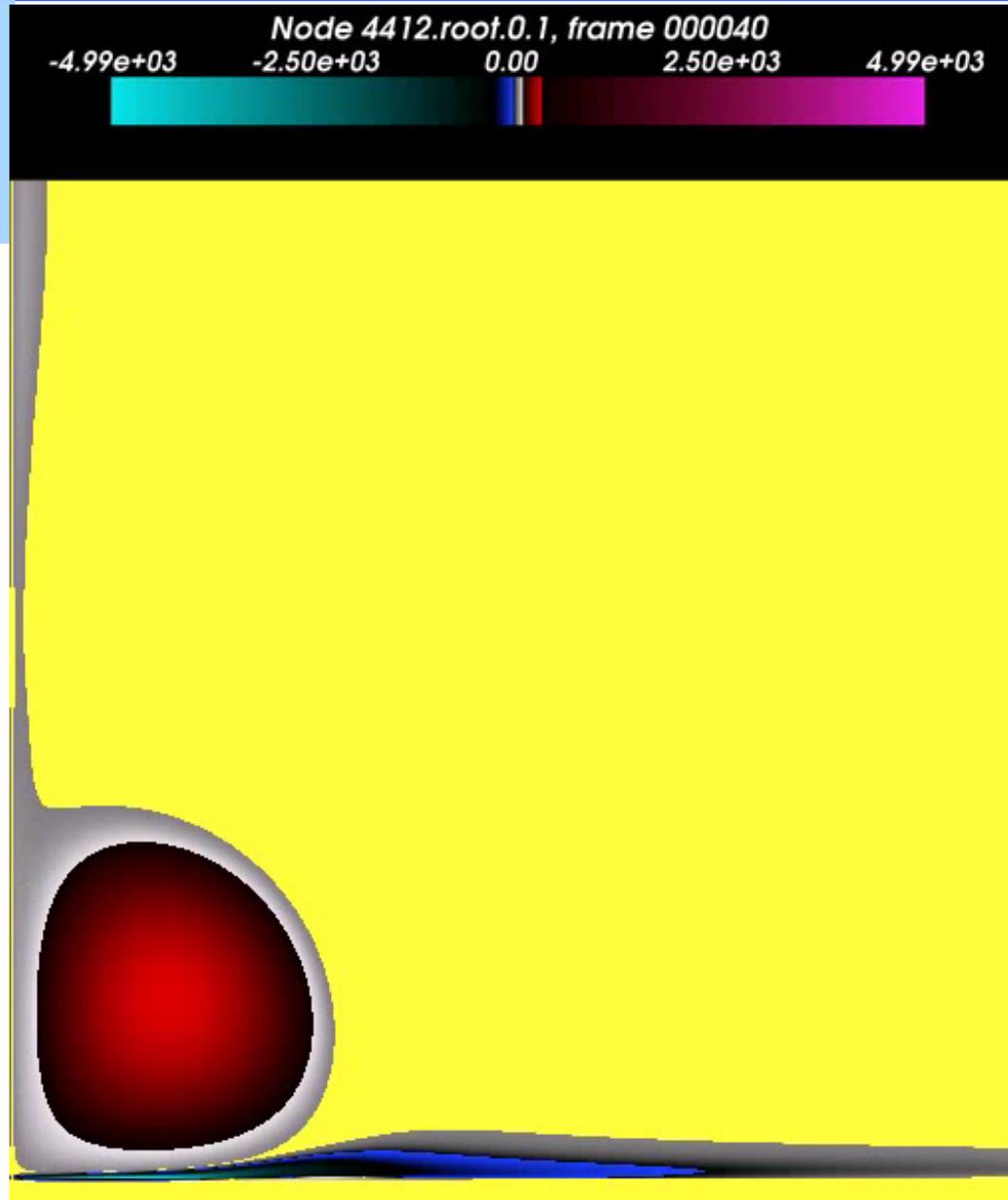
$t=0.3$

$t=0.4$

$t=0.5$

Dipole-wall collision at $Re=8000$

DNS
Resolution
 $N=8192^2$

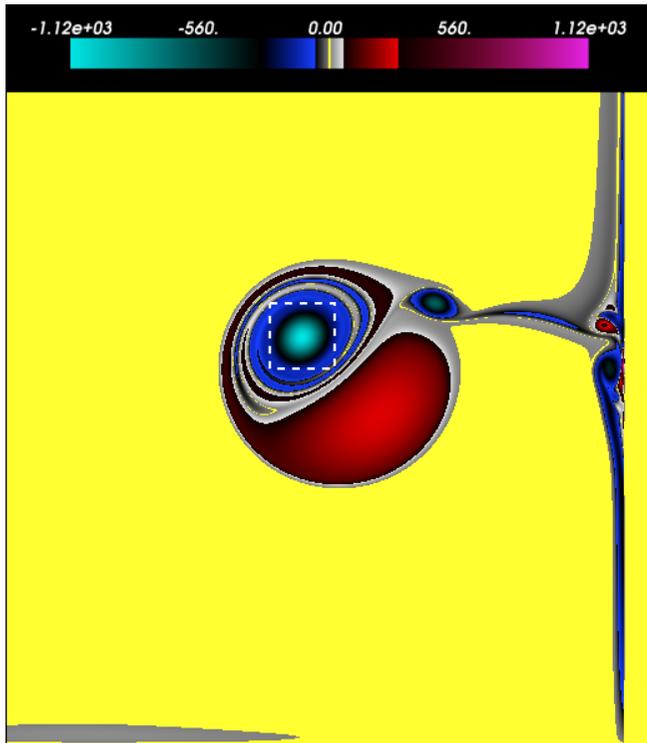


Production of vortices where boundary layer detaches

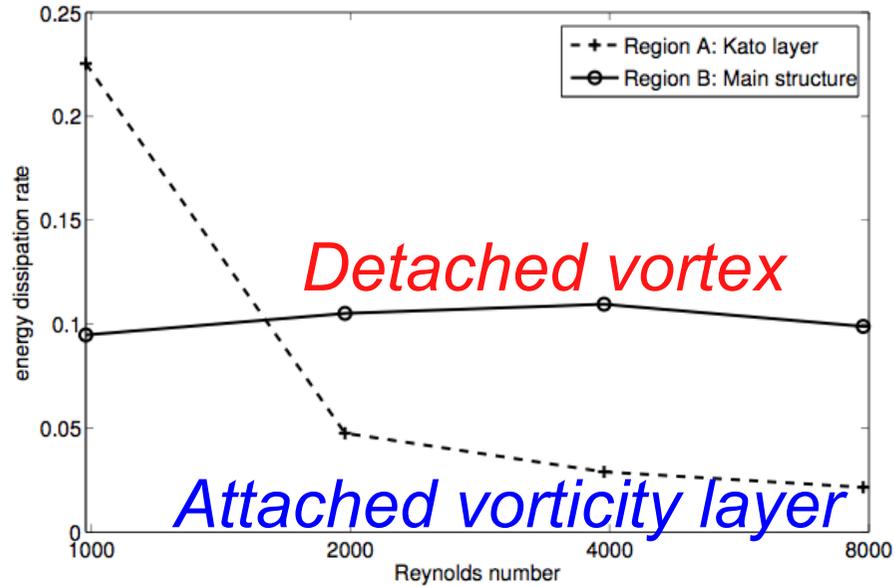
Nguyen van yen, M. F.
and Schneider,
PRL, 106(18)

Dissipative structures

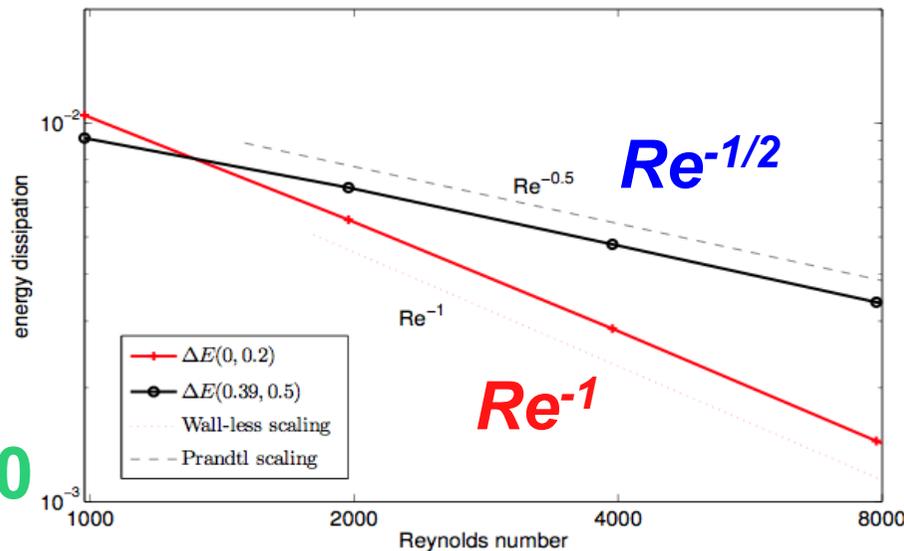
Nguyen van yen, M. F.
and Schneider,
PRL, 106(18)



Re=1000



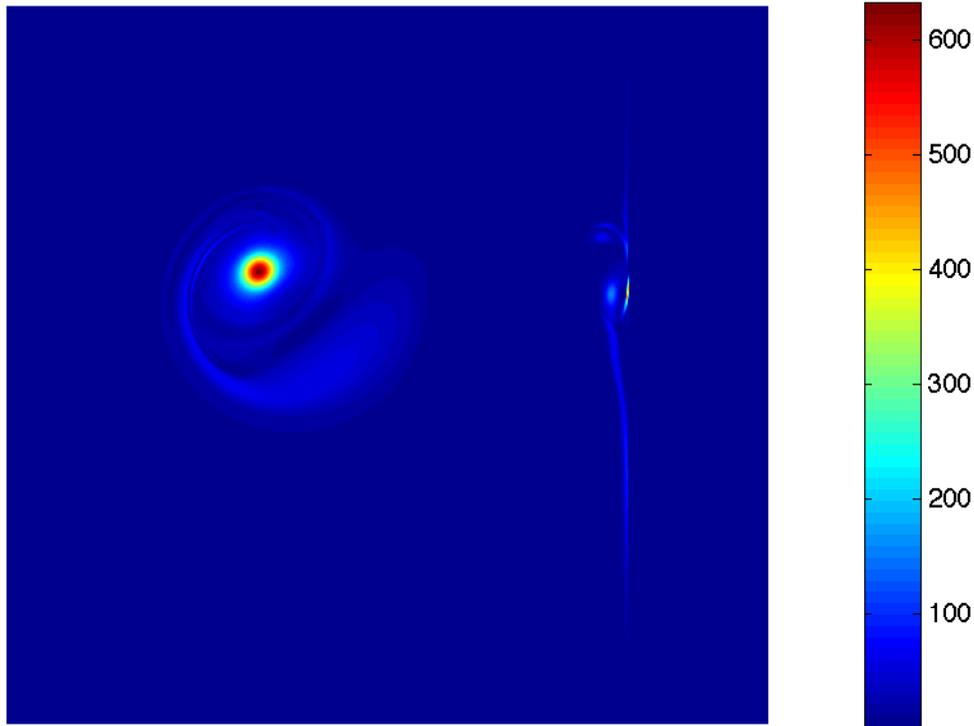
energy
dissipation
rate



energy
dissipation

Re=8000

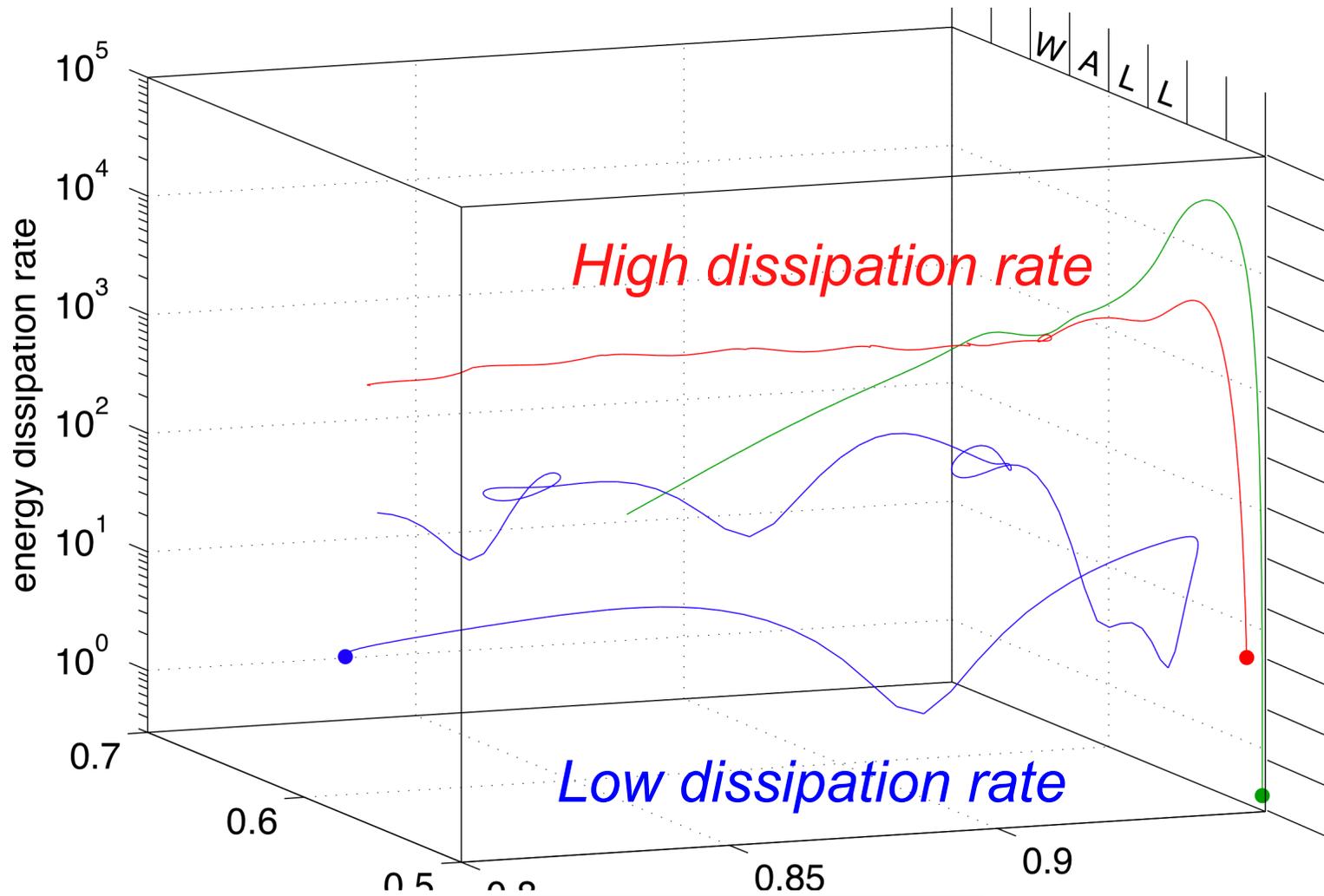
Snapshot of the local dissipation rate



The strongest values of the energy dissipation rate is observed inside the main vortex that detached from the boundary layer, rather than inside the boundary layer itself.

*Local dissipation rate
for the dipole-wall collision
at $t= 0.5$*

Production of dissipative structures



R. Nguyen van yen, M. F.
and K. Schneider,
PRL, 106(18)

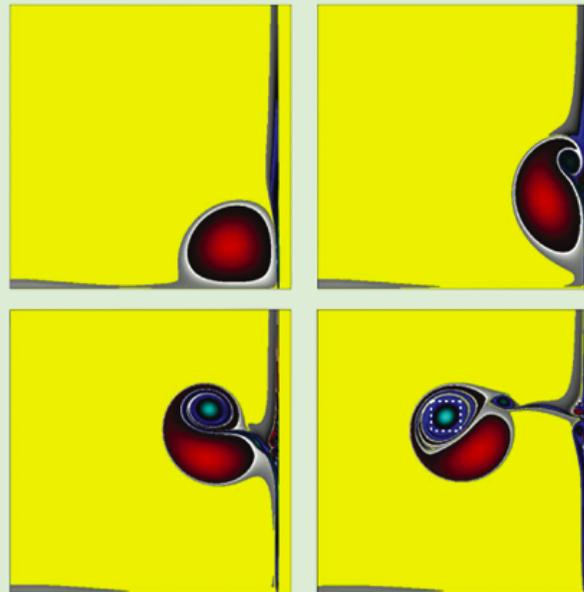
Energy Dissipating Structures Produced by Walls in Two-Dimensional Flows at Vanishing Viscosity

R. Nguyen van yen, M. F. and K. Schneider, 2011

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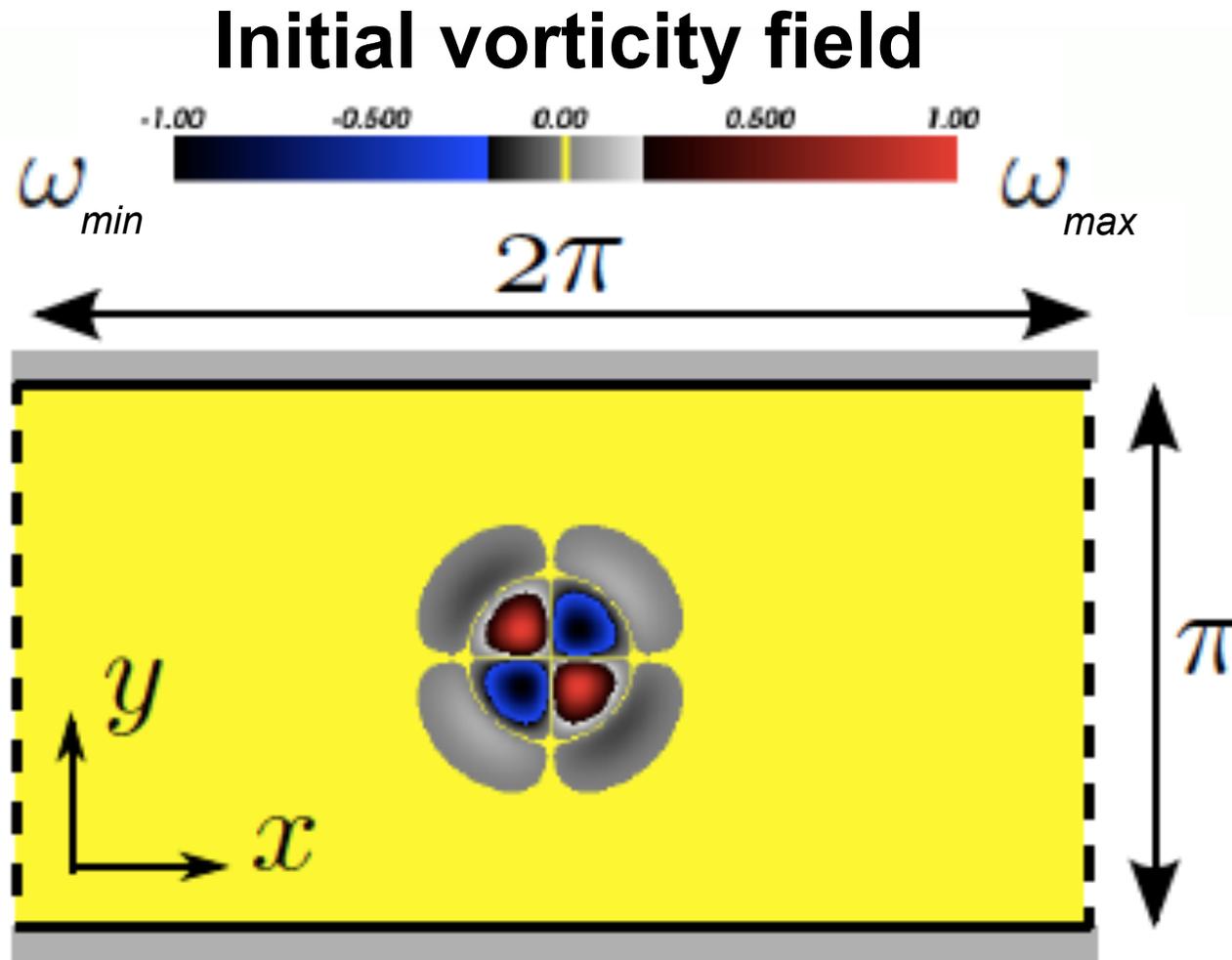


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physics

Volume 106, Number 18

33. Quadrupole in a channel flow



$$\psi_i(x, y) = Axy \exp\left(-\frac{(x - x_0)^2 + (y - y_0)^2}{2s^2}\right)$$

Prandtl equations

Ansatz for the vorticity field as $Re \rightarrow \infty$:

$$\omega(x, y) = \omega_E(x, y) + \nu^{-1/2} \omega_P(x, \nu^{-1/2} y) + \omega_R(x, y)$$

$$\mathbf{y}_P = \mathbf{y} / \nu^{1/2}$$

$$\partial_t \omega_P + \nabla \cdot (\mathbf{u}_P \omega_P) = \partial_{y_P}^2 \omega_P$$

$$\omega_P(x, y_P, 0) = 0$$

$$\psi_P(x, y_P, t) = \int_0^{y_P} dy'_P \int_0^{y'_P} dy''_P \omega_P(x, y''_P, t)$$

$$\partial_{y_P} \omega_P(x, 0, t) = -\partial_x p_E(x, 0, t),$$

where p_E is the pressure calculated from ω_E
which is the vorticity given by Euler equation

Prandtl solver

- Artificial boundary condition: $\partial_{y_P} \omega_P = 0$ at $y_P = 64$
- Spatial discretization: **Fourier** in x
and **compact finite differences of 5th order** in y
- Time discretization: low storage **third order Runge-Kutta** in t

- **Neumann boundary condition for vorticity:**

$$\partial_{y_P} \omega_P = -\partial_x p_E \text{ at } y_P = 0$$

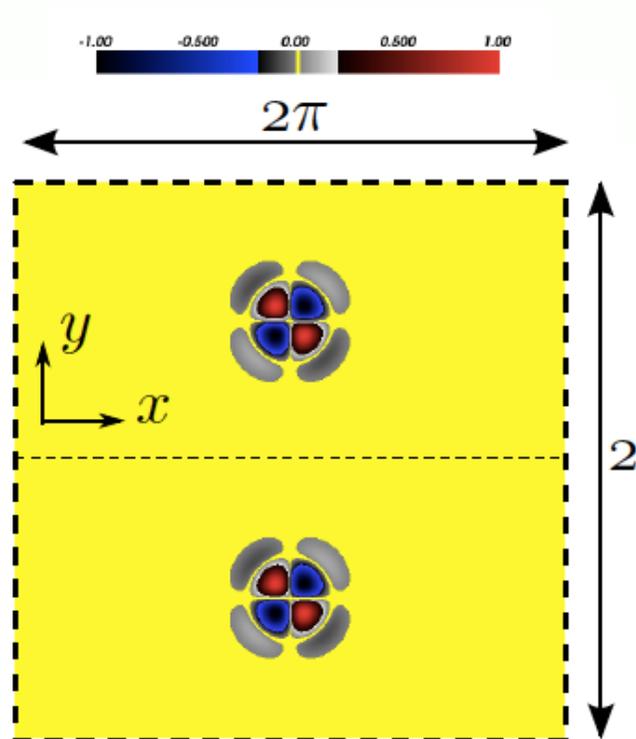
where p_E is the pressure calculated from ω_E

- To close the system we impose

$$\partial_{y_P}^2 \omega_P = 0 \text{ at } y_P = 64$$

which is consistent with the rapid decay of ω_P

Euler solver

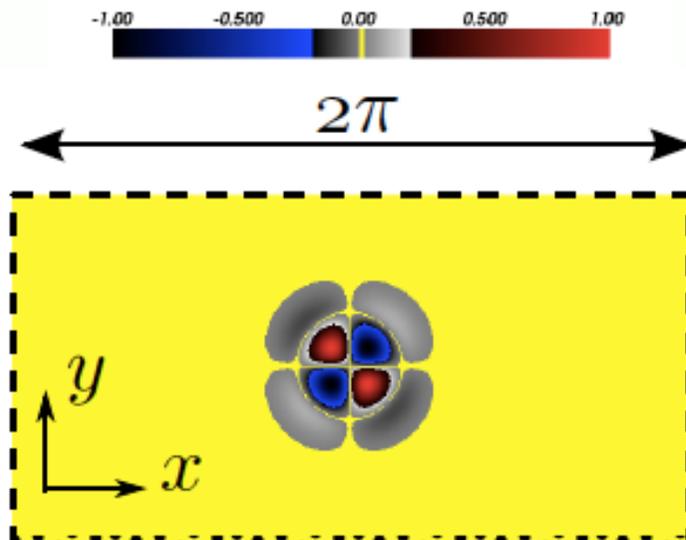


- Use mirror symmetry around $y = 0$ to impose boundary condition.
- Spatial discretization: Fourier pseudo-spectral with hyperdissipation, $k_{max} = 682$.
- Time discretization: third order low storage Runge-Kutta, with exponential propagator for the viscous term.

Navier-Stokes solver

Fourier/compact finite differences (5th order)

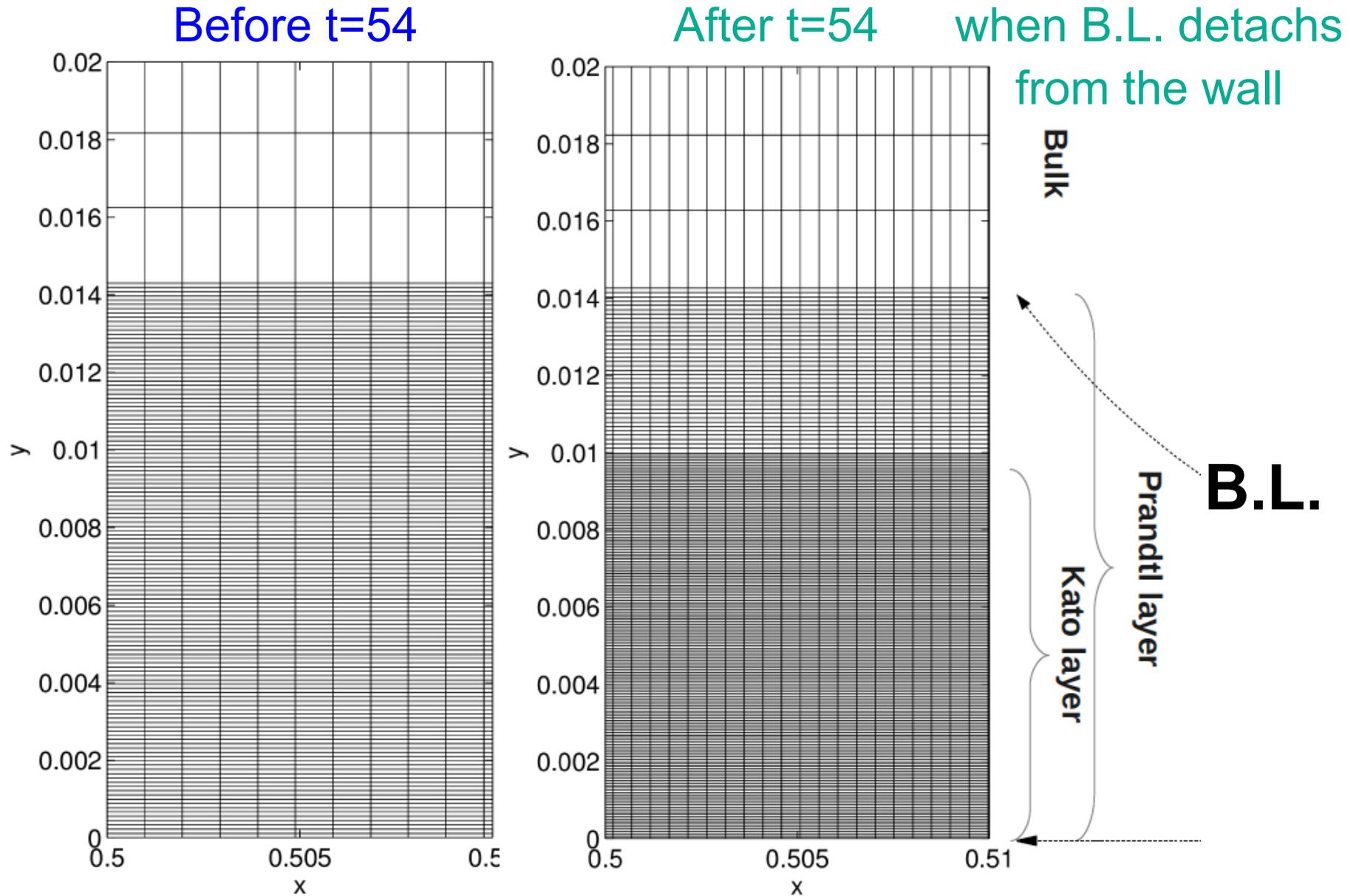
- Similar to the one for the Prandtl equations, except that non-uniform grids are used in the y direction.
- Two linear integral constraints are applied on vorticity to satisfy the no-slip boundary conditions in y .
- Integrating factor for the viscous term and 3rd order Runge-Kutta for the advection term.

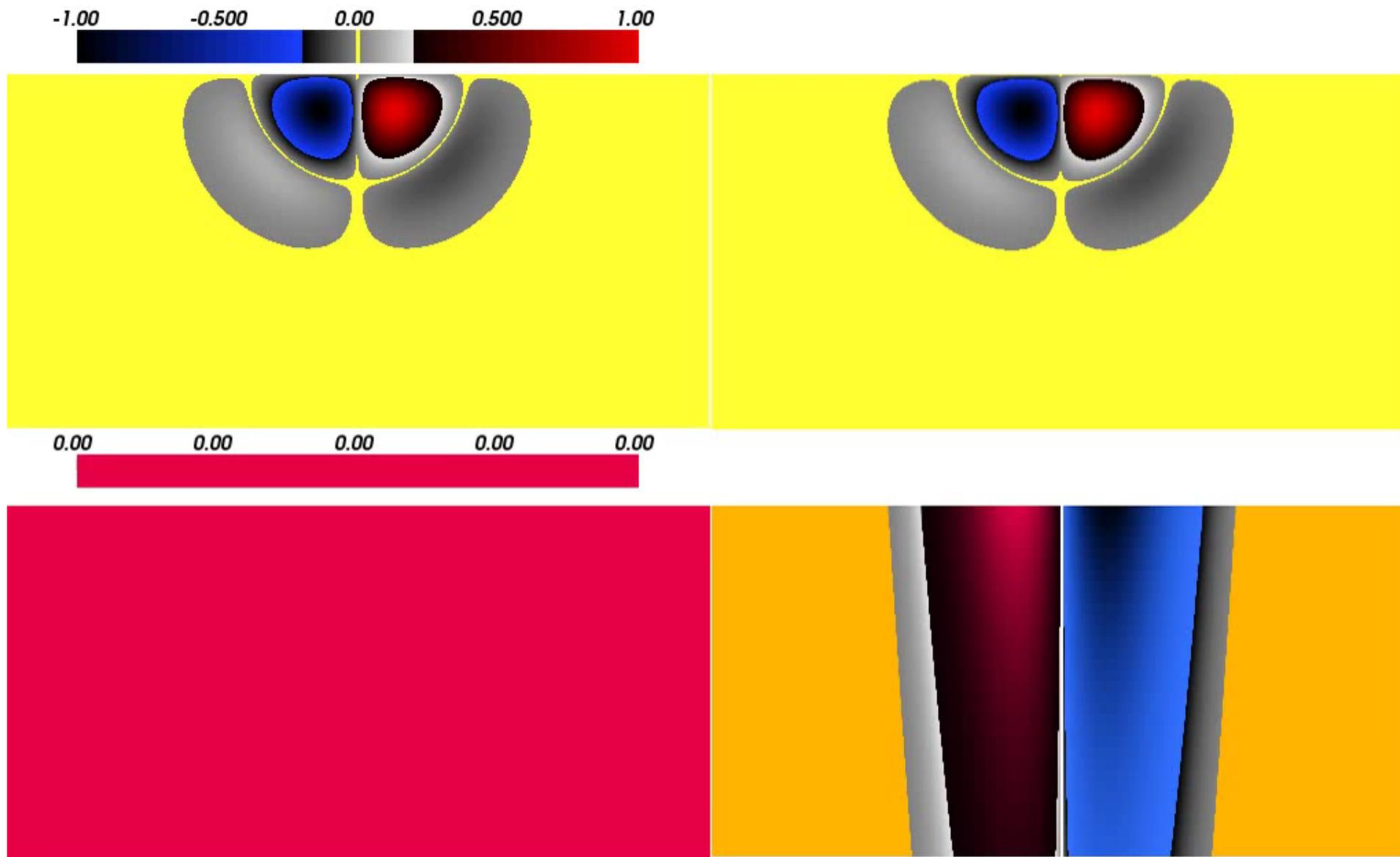


$$N_x = 1024$$

$$N_y = 385 - 449$$

Computational grid





Euler Prandtl couplées

Navier-Stokes

Prandtl's singularity

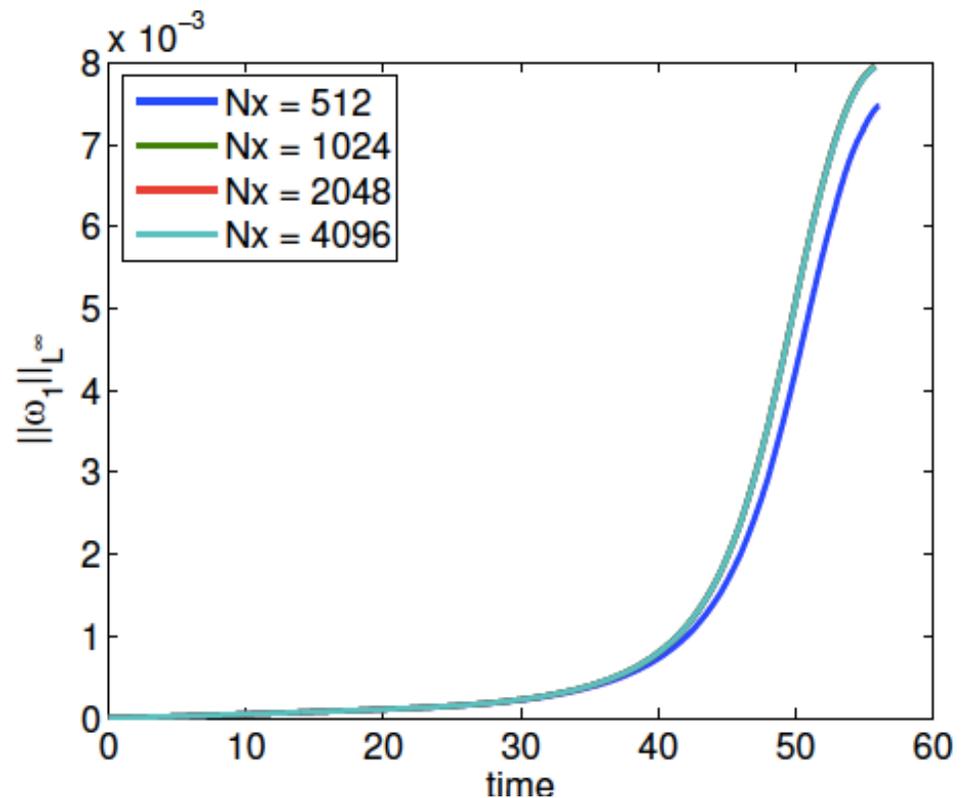
Prandtl equation has well-known finite time singularity

- $|\partial_x \omega_1|$ and $u_{1,y}$ blows up,
- ω_1 remains bounded.

L. L. van Dommelen
and S. F. Shen., 1980
J. Comp. Phys., **38**(2)

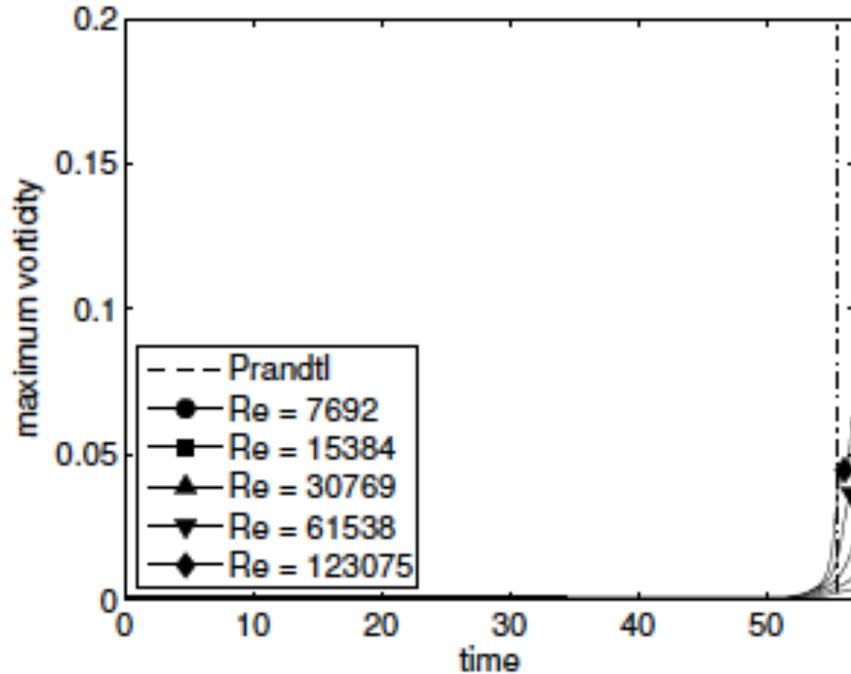
This is observed
in our computations
as expected,

for $t \rightarrow t_D \simeq 55.8$

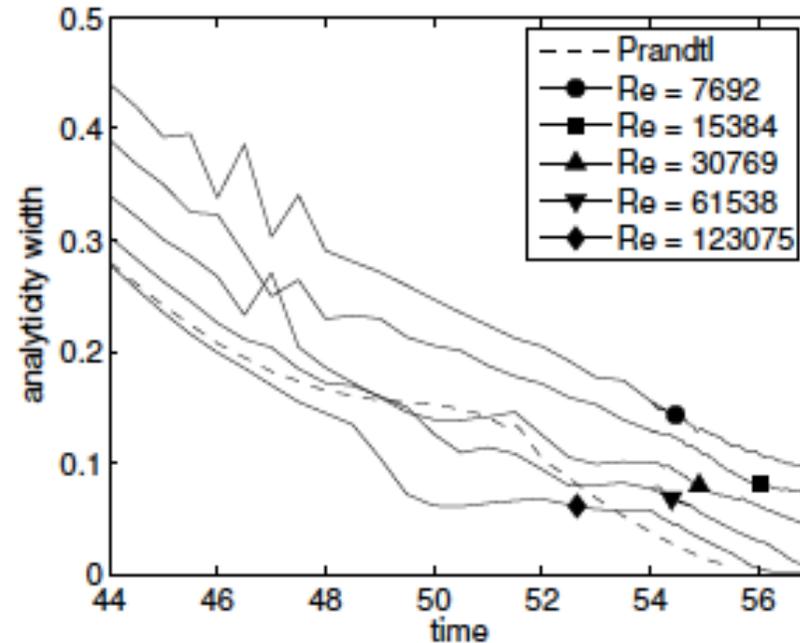


Prandtl solution's blow-up

According to Kato's theorem, and since ω_1 remains bounded uniformly until t_D , we expect that $\mathbf{u}_\nu \xrightarrow[\nu \rightarrow 0]{L^2} \mathbf{u}_0$ uniformly on $[0, t_D]$.



Evolution of vorticity max

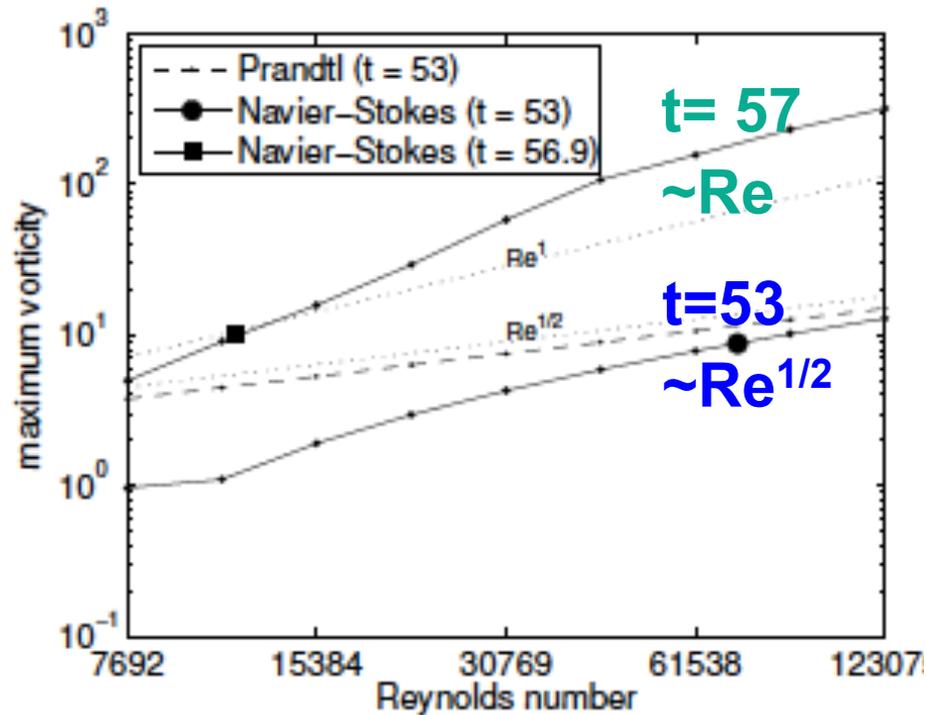


Evolution of analyticity strip

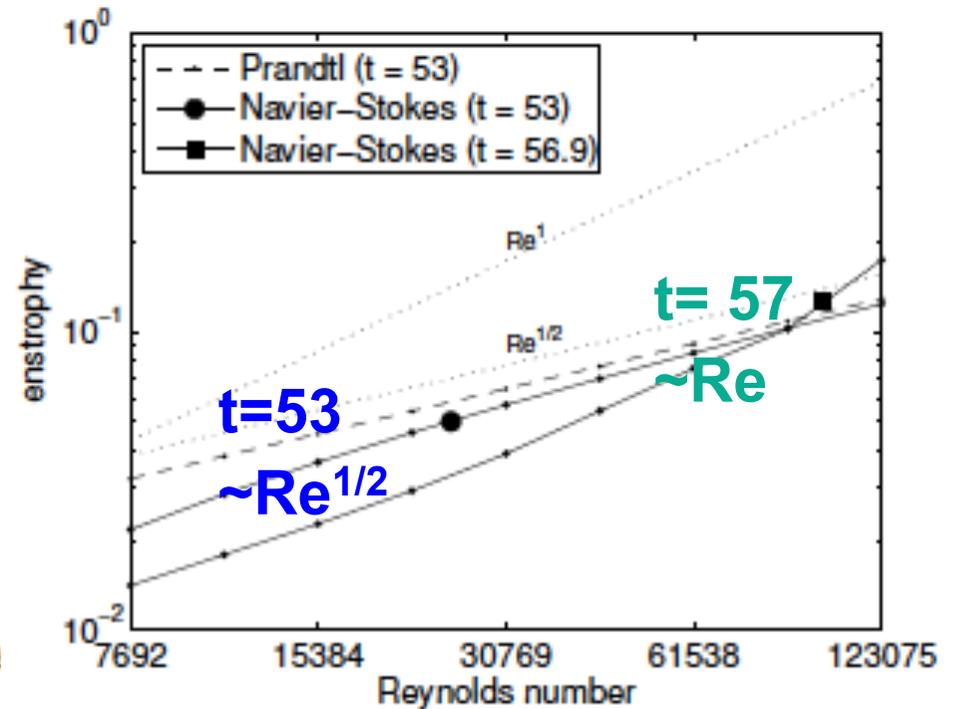
Show convergence!

What happens after the singularity?

Vorticity max

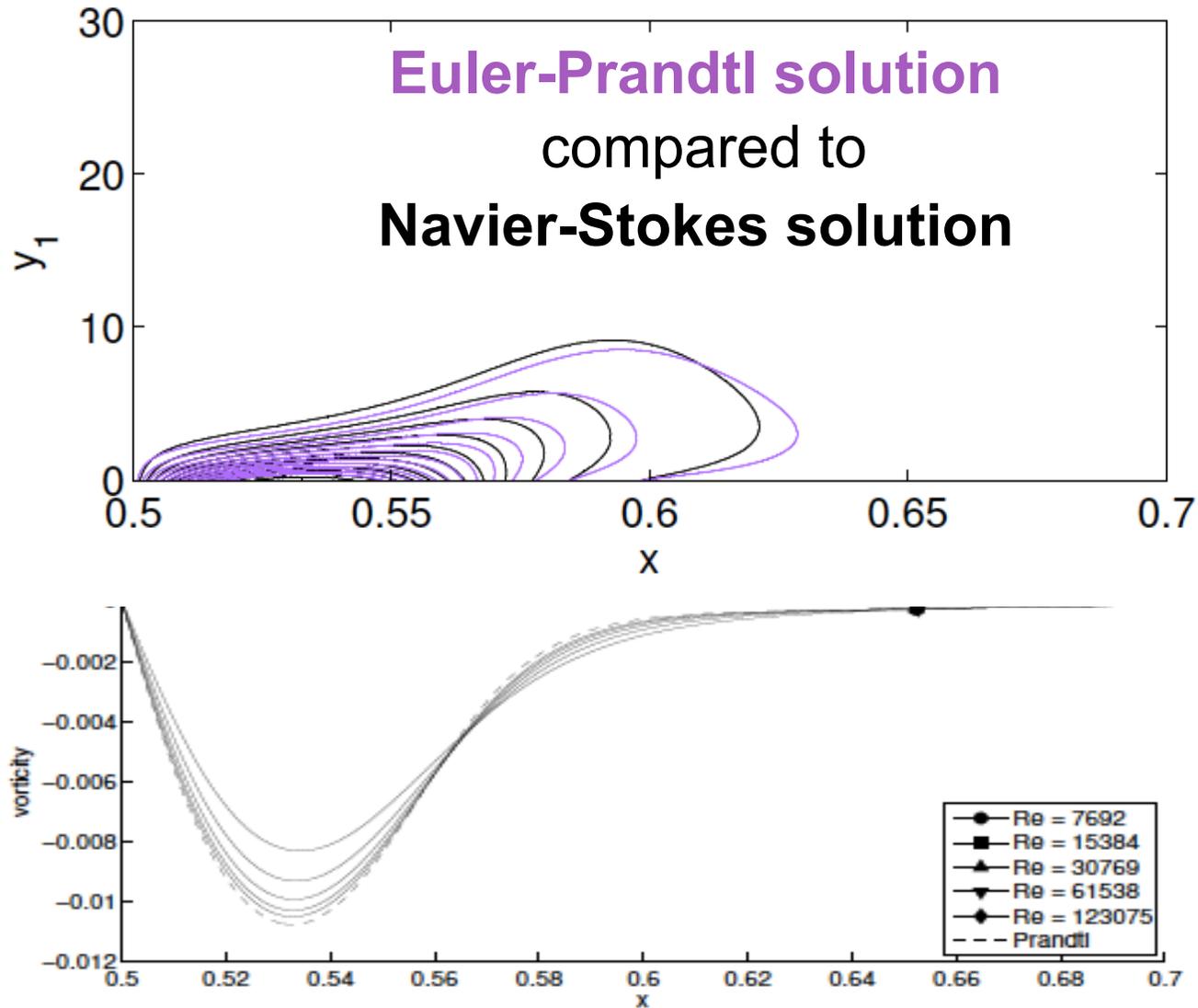


Enstrophy

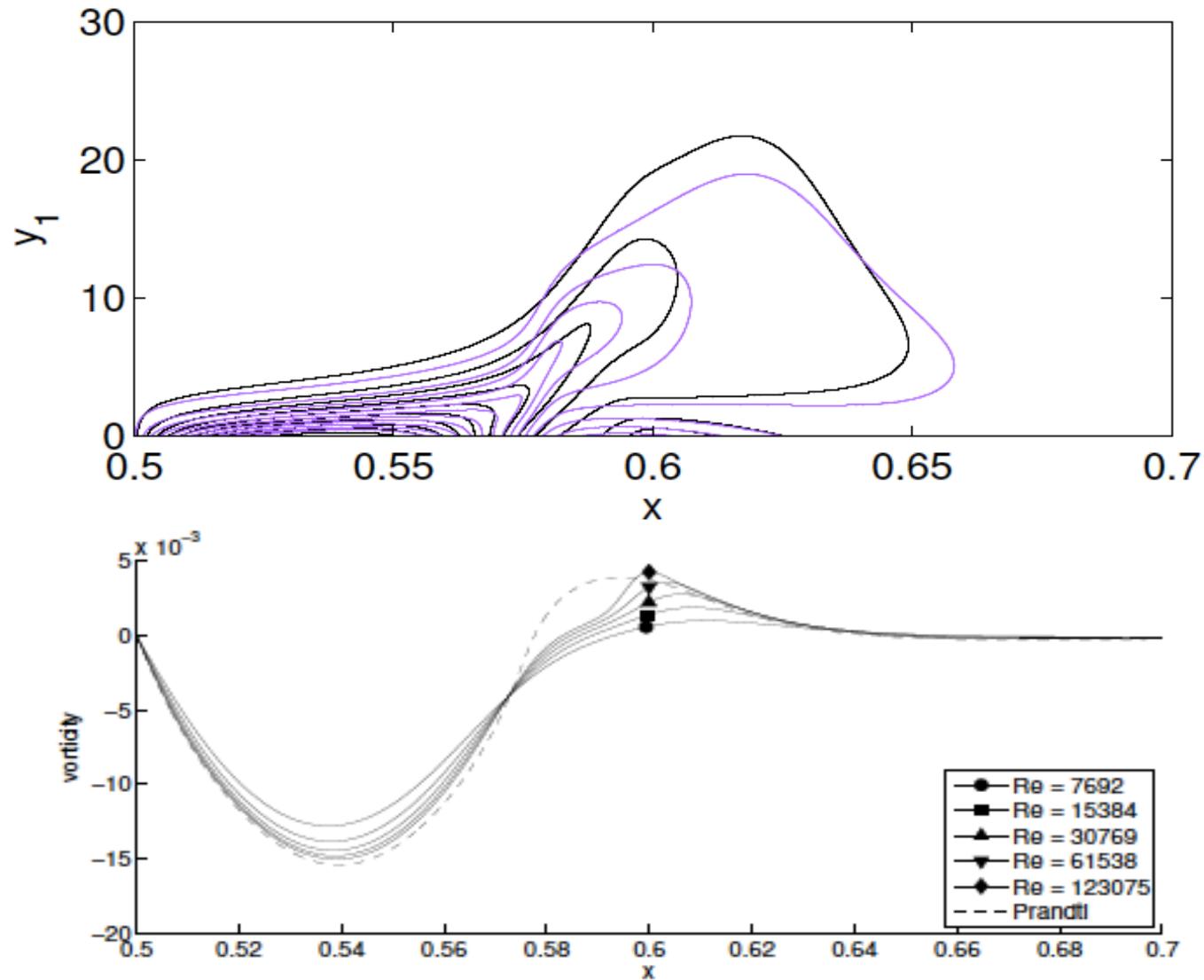


We observe Prandtl's scaling in $Re^{1/2}$ before $t_D \sim 55.8$
and Kato's scaling in Re after

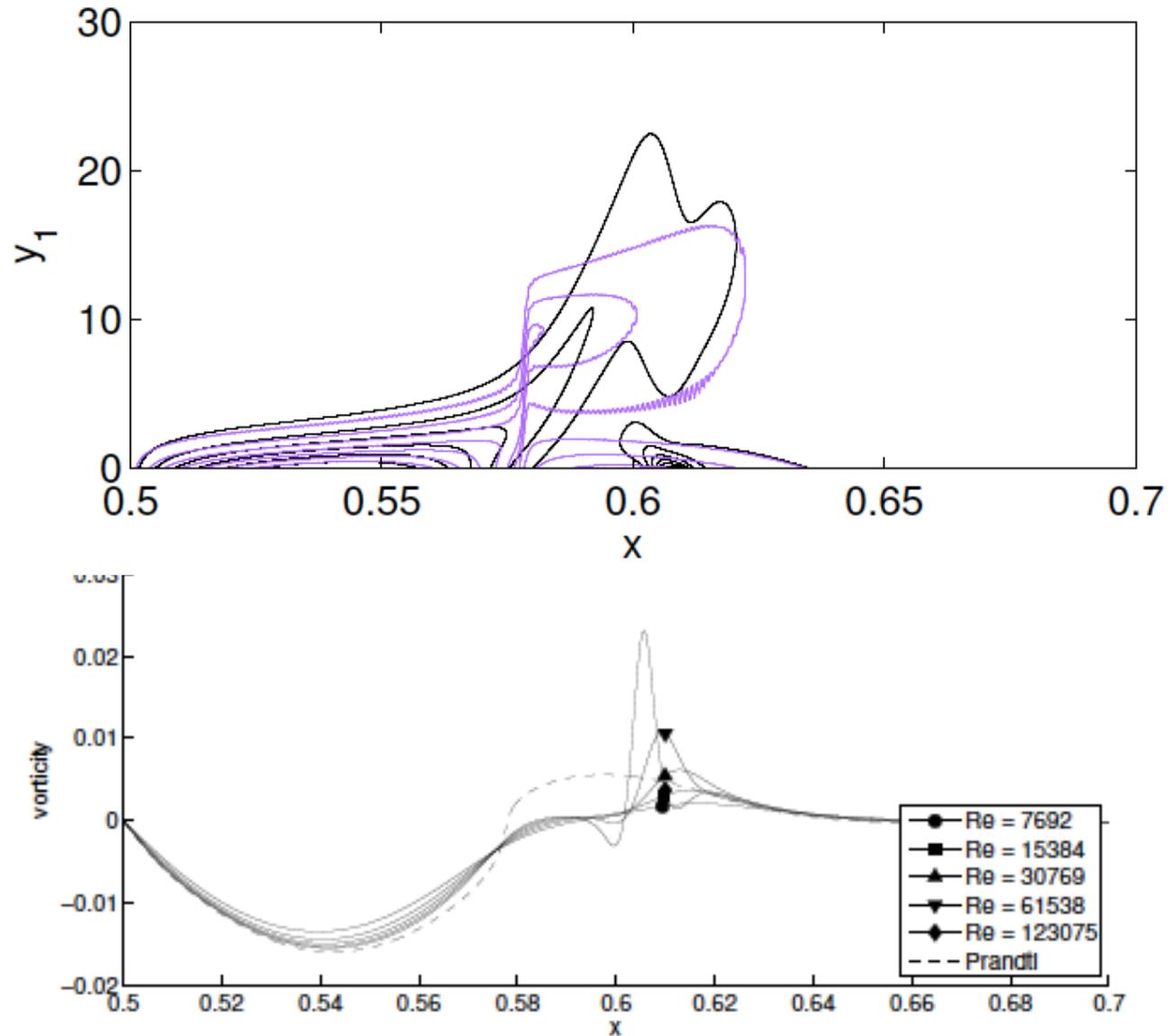
Vorticity along the wall at t=50



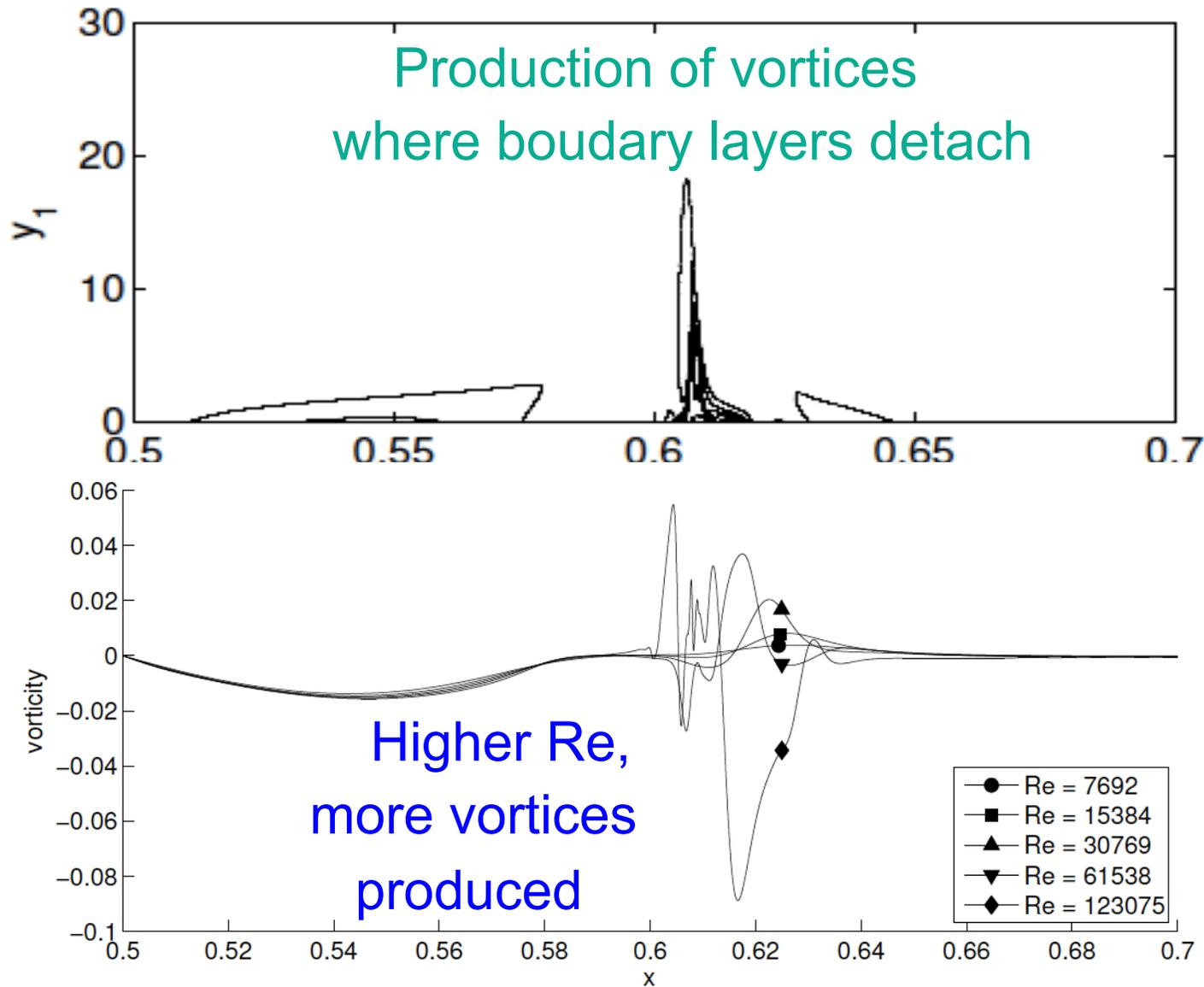
Vorticity along the wall at t=54



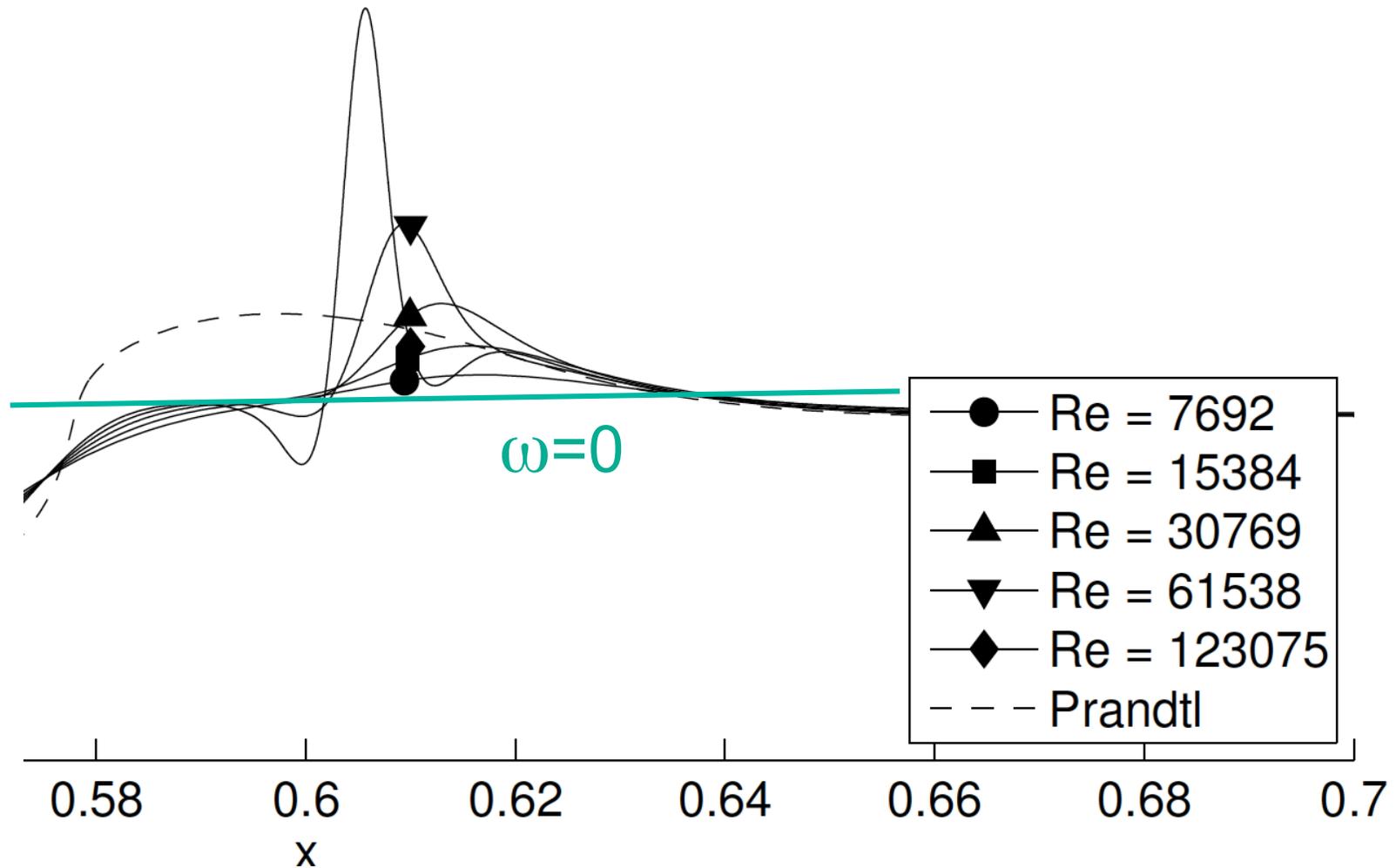
Vorticity along the wall at t=55



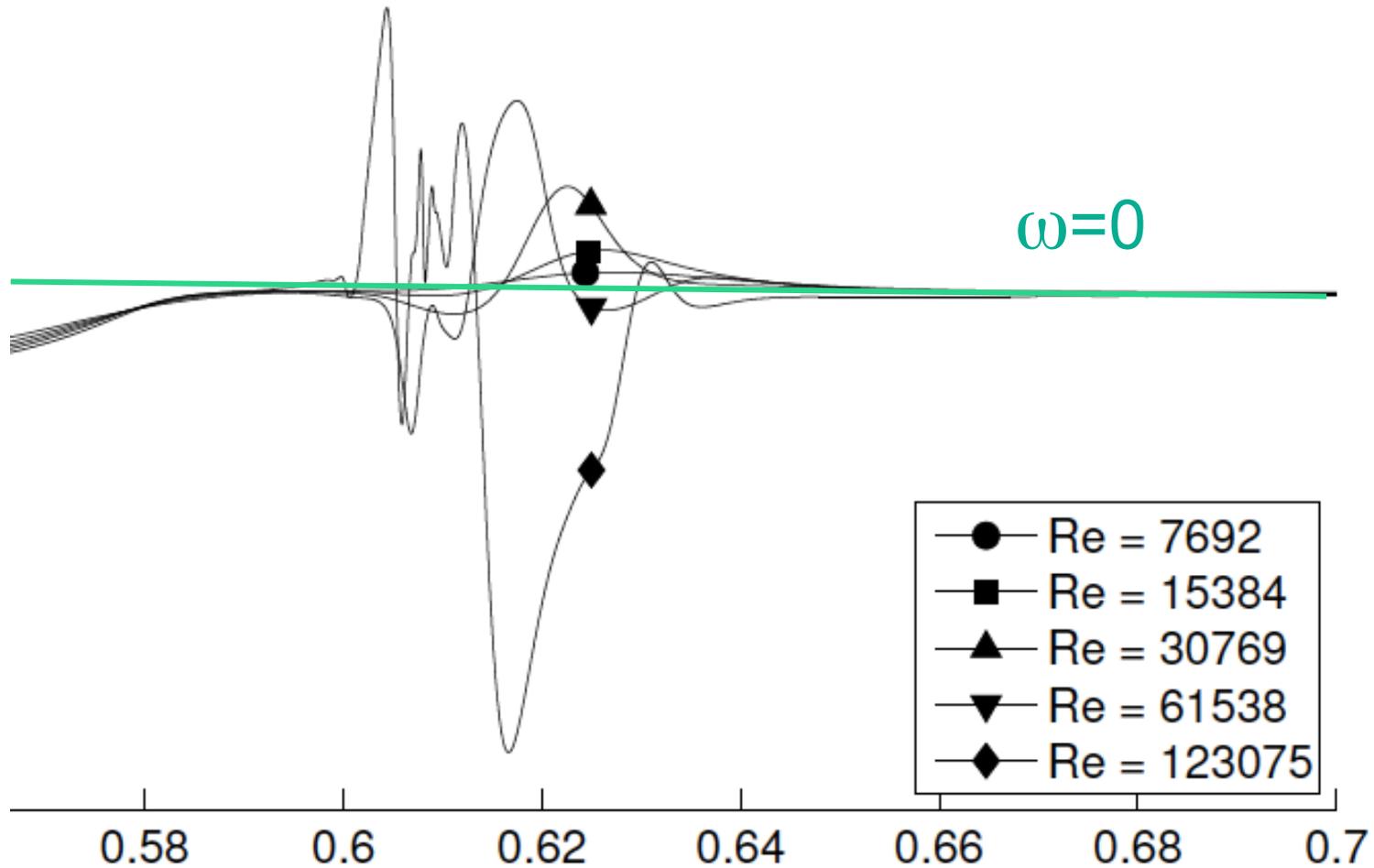
Vorticity along the wall at $t=57 > t_D$



Vorticity along the wall at t=55.3



Vorticity along the wall at t=57.5



Conclusion

The production of dissipative structures is the key feature of boundary layer (BL) detachment at vanishing viscosity limit of incompressible flows.

The viscous Prandtl solution becomes singular at t_D That corresponds to the instant when BL detaches.

The viscous Navier-Stokes solution converges uniformly to the inviscid Euler solution for $t < t_D$, and ceases to converge for $t > t_D$.

The detachment process involves spatial scales in different directions, and not only parallel to the wall, that are as fine as Re^{-1} .

Conclusion

The Navier-Stokes boundary layer detachment dynamics are very different from the dynamics of the finite time singularity developing in Prandtl's equation with:

- non locality in the parallel direction,
- formation of small scales scaling at least as Re^{-1} , in different directions and not only in the direction parallel to the wall,
- pressure plays an essential role in the detachment process.

*R. Nguyen van yen, M. F. and
K. Schneider, 2011
Phys. Rev. Lett., 106(18), 184502*

*R. Nguyen van yen, M. Waidman, R. Klein,
M.F. and K. Schneider, 2014
Preprint*

Open questions

Numerical results suggest that a **new asymptotic description of the flow beyond the breakdown** of the Prandtl regime is possible.

Studying it might help to answer the following questions:

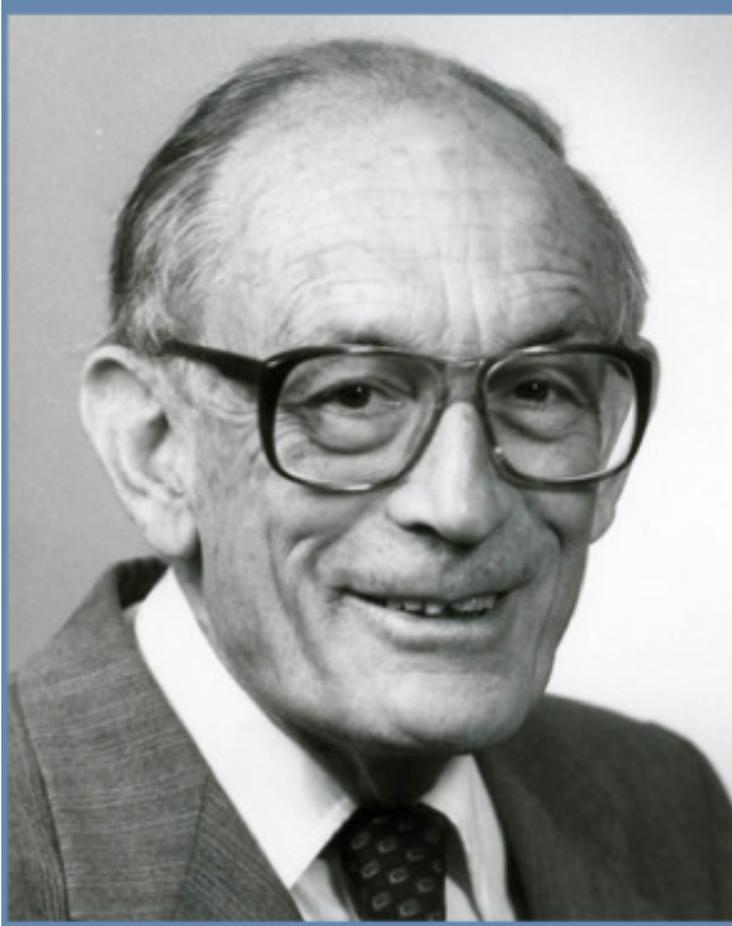
- **Would Navier-Stokes solution loses smoothness** after t_D ?
- Would it **converges to a weak singular dissipative solution of Euler's equation** analog to dissipative shocks in Burgers solution?
- **How can such a weak solution be approximated numerically?**

This might lead to a **new resolution of d'Alembert's paradox** in terms of the **production of weak singular dissipative structures** due to the interaction of fully-developed turbulent flows with walls.

*J. Leray, 1934
Sur le mouvement d'un fluide visqueux,
Acta Mathematica, 63*

*C. de Lellis and L. Székelyhidi, 2010
Archives Rational Mechanics and Analysis,
195(1), 221-260*

On turbulence

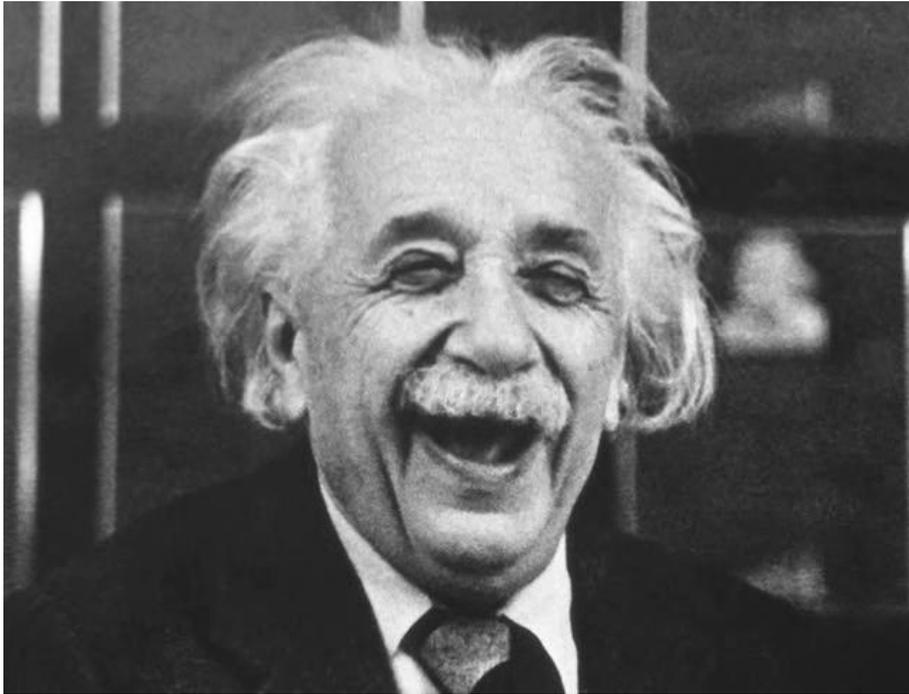


*Hans Liepmann
(1914-2009)*

'As long as we are not able to predict the drag on a sphere or the pressure drop in a pipe from continuous, incompressible and Newtonian assumptions without any other complications, namely from first principles, we would not have made it!'

*Turbulence Workshop
UC Santa Barbara
1997*

On mathematics and reality



*Albert Einstein
(1879-1955)*

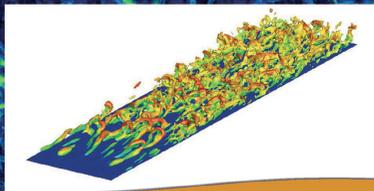
'As far as the laws of mathematics refer to reality, they are not certain, as far as they are certain, they do not refer to reality'.

*'Geometry and experience',
Conference given in Berlin
at the Prussian
Academy of Sciences
on January 27th 1921*

Turbulence Colloquium Marseille TCM2011

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